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ANALYSIS OF A COMPRESSIBLE FLUID SOFT RECOIL (CFSR) CONCEPT APPLIED TO A 155 MM HOWITZER

BJORN L. HOFGAARD

MARCH 1979



US ARMY ARMAMENT RESEARCH AND DEVELOPMENT COMMAND
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20. ABSTRACT (Continue on reverse side if necessary and identify by block number) A mathematical model analyzes a compressible fluid soft recoil (CFSR) concept applied to a 155 mm howitzer. The model addresses the relationships between volume and pressures in a compressible fluid, the forces, deflections, and stresses in the walls of a cylinder which is designed to expand elastically, and the forces and velocities of the recoiling mass. The mathematical model is used with a computer to optimize sizes, pressures, and stresses of the recoil mechanism.		

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INTRODUCTION

Attempts are being made to simplify soft recoil artillery mechanisms, to increase their reliability, and to reduce their need for costly and time-consuming maintenance and rebuild programs.

One approach proposed to accomplish these ends consists of storing the needed operating power or run-up energy partially in an elastically-compressed fluid and partially in elastically-expanding cylinder walls of the recoil mechanism itself. This compressible fluid soft recoil (CFSR) mechanism contains no mechanical springs, gas, replenishers, or recuperators. This results in tremendous simplification. The feasibility of this concept is analyzed in this technical report.

Not much work has been done in this field. Consequently, there are a multitude of unknown parameters and only a limited number of independent physical phenomena taking place which can be set up in mathematical form to solve these unknowns. The unknown parameters include: diameters, lengths, shapes, thicknesses, and orifice sizes; constants, coefficients, physical relationships, and physical properties; pressures, forces, pressure areas, and stresses; fluid flows, volumes, and weights; velocities, required energies, etc. Some of the less critical parameters, therefore, had to be arbitrarily estimated, and some variables had to be fixed.

The basic CFSR concept is thoroughly explained in reference 1. In addition, a brief description is included in the following paragraphs.

In a soft recoil mechanism, a force causes the recoiling mass to move forward before the round is fired. Firing is initiated when a predetermined level of kinetic energy is reached, which is about half of the energy transmitted to the breech by the firing forces. By the time the recoiling mass has returned to the latch position and restored the working energy, all of the firing energy has been dissipated without any large forces being transmitted to the rest of the structure. This permits the size and weight of the remaining structure to be greatly reduced.

In the CFSR concept described by this report, only part of the force which sends the recoiling mass forward before firing is derived from a liquid which has been elastically compressed.

An additional force is imparted by the outer walls of the recoil cylinder which is designed to expand under pressure and store energy elastically.

The expanding cylinder wall theory is a new concept which is evaluated in conjunction with the basic CFSR principle.

DEVELOPMENT AND THEORY OF THE MATHEMATICAL MODEL

A mathematical model is used primarily to identify and measure significant relationships between controlling parameters. When developing a model, only the most important variables are considered and any secondary effects are generally neglected. Inclusion of too much detail tends to obscure the important relationships, to reduce ability to examine sensitivity to varying input data, and to increase the time required for model development.

In the following paragraphs the most important parameters are optimized (fig. 1, 2).

Determination of Cylinder Expansion

For a thick-walled cylinder with capped ends, having inner and outer radii of r_1 and r_2 , respectively, and subjected to an internal pressure, P_1 , and an external pressure, P_2 , the tangential, longitudinal, and radial stresses are defined, respectively, by the following equations (ref. 2):

$$\sigma_t = \frac{P_1 r_1^2 - P_2 r_2^2 + (r_1^2 r_2^2 / \rho^2)(P_1 - P_2)}{r_2^2 - r_1^2}, \quad (1)$$

$$\sigma_l = \frac{P_1 r_1^2 - P_2 r_2^2}{r_2^2 - r_1^2}, \quad (2)$$

$$\sigma_r = \frac{P_2 r_2^2 - P_1 r_1^2 + (r_2^2 r_1^2 / \rho^2)(P_1 - P_2)}{r_2^2 - r_1^2}, \quad (3)$$

where p is the radius to an arbitrary element of the cylinder. The strain in the cylinder is given as

$$\epsilon_t = \frac{1}{E} (\sigma_t - \nu \sigma_r - \nu \sigma_\ell) \quad (4)$$

(stresses are positive in tension) where ν is Poissons's ratio (0.287) and E is the modulus of elasticity 199.95×10^9 Pa (29×10^6 psi).

For a thick-walled cylinder with capped ends, having inner and outer radii of a and b , respectively, and subjected to an internal pressure only, then; $P_1 = P$, $r_1 = a$, $r_2 = b$ and $P_2 = 0$.

For $\rho = r_1 = a$

$$\sigma_t = \frac{b^2 + a^2}{b^2 - a^2} P \quad \text{tension,} \quad (5)$$

$$\sigma_\ell = \frac{a^2}{b^2 - a^2} P \quad \text{tension,} \quad (6)$$

$$\sigma_r = P \quad \text{compression,} \quad (7)$$

$$\epsilon = \frac{\Delta a}{a} = \frac{1}{E} \left[\frac{b^2 + a^2}{b^2 - a^2} P + \nu P - \nu \frac{a^2}{b^2 - a^2} P \right], \quad (8)$$

so

$$\Delta a = P \frac{a}{E} \left[\frac{b^2 + (1 - \nu)a^2}{b^2 - a^2} + \nu \right]. \quad (9)$$

For a thick-walled cylinder with capped ends, having inner and outer radii of c and d , respectively, and subjected to an external pressure only, then; $P_1 = 0$, $P_2 = P$, $r_1 = c$, $r_2 = d$.

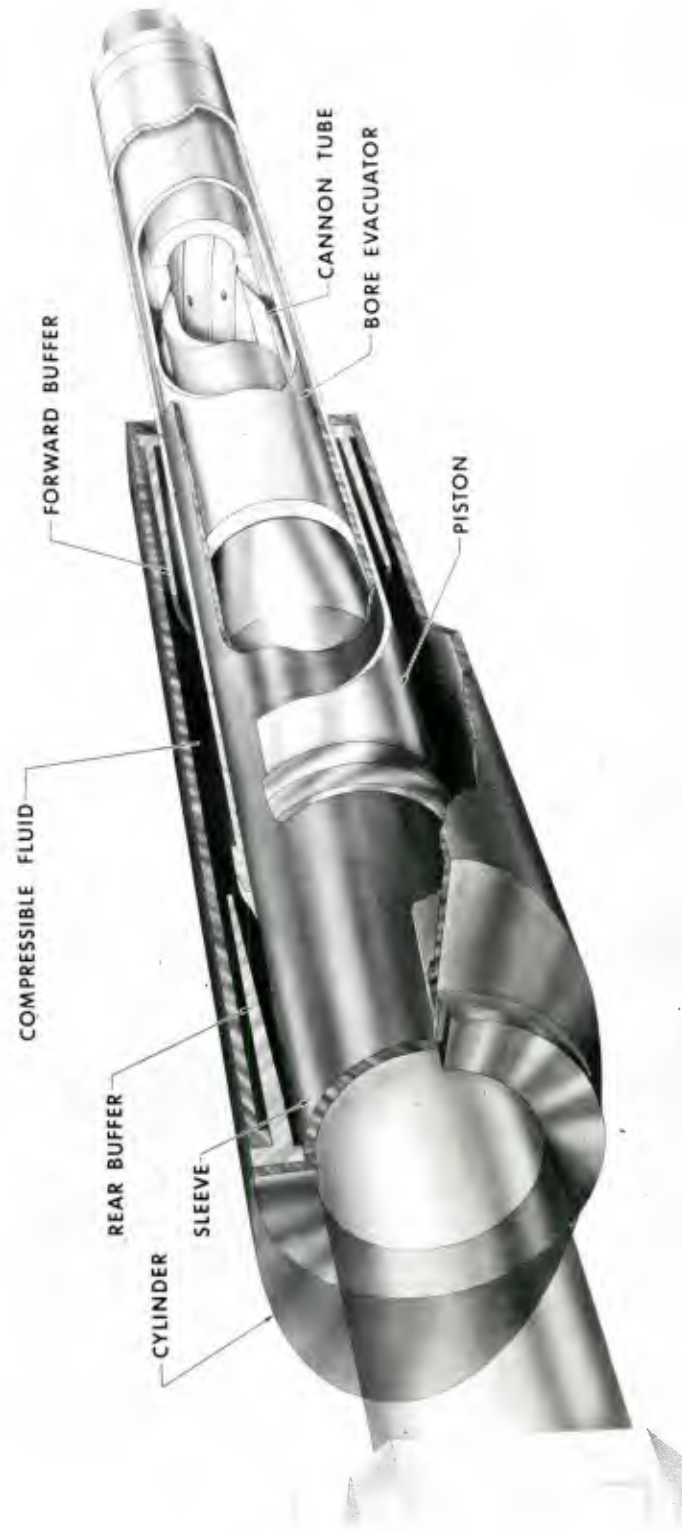


Figure 1. Compressible fluid soft recoil concept.

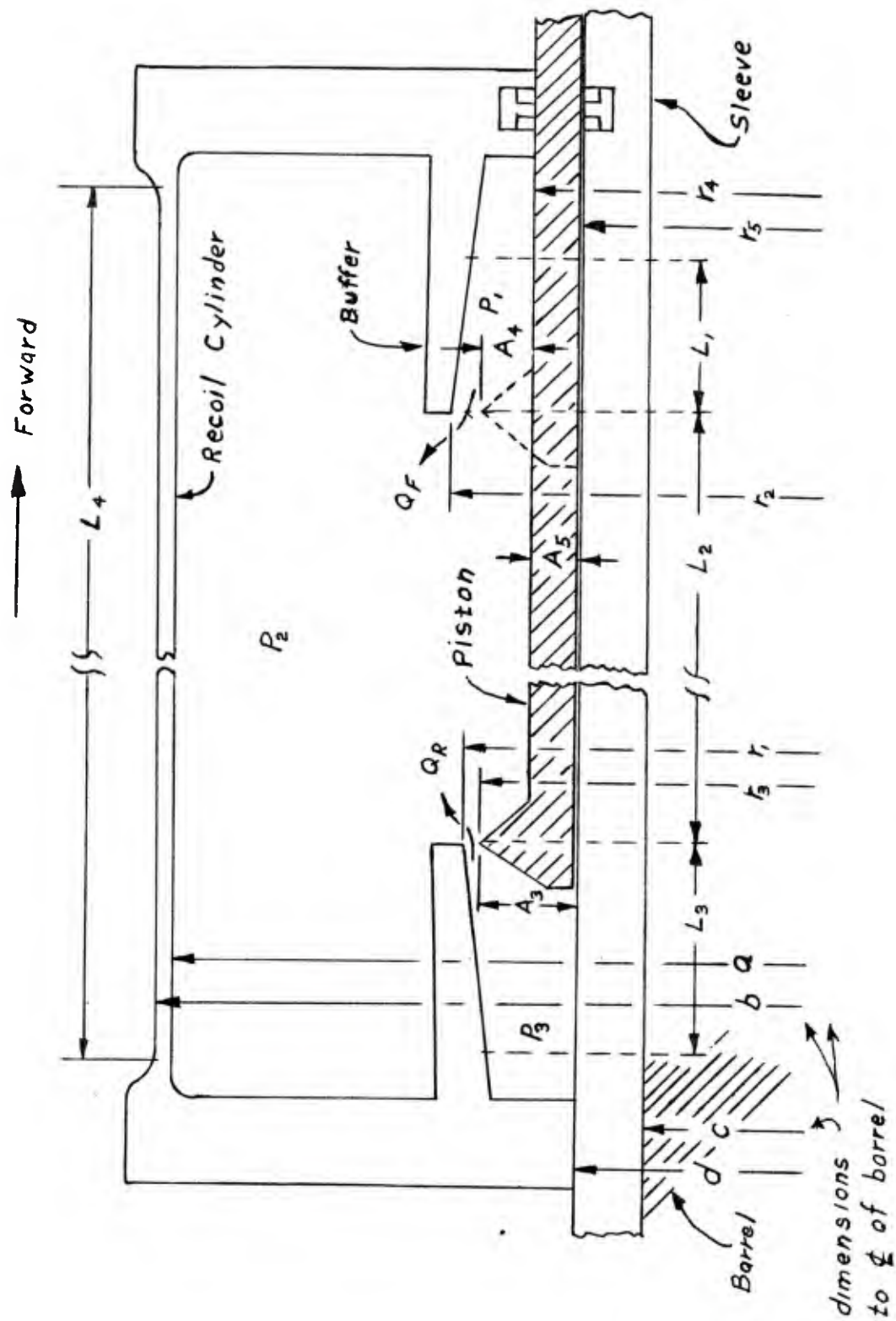


Figure 2. Schematic half section of recoil mechanism.

For $\rho = r_2 = d$

$$\sigma_t = -p \frac{d^2+c^2}{d^2-c^2} \quad \text{tension,} \quad (10)$$

$$\sigma_\ell = -p \frac{d^2}{d^2-c^2} \quad \text{tension,} \quad (11)$$

$$\sigma_r = p \quad \text{compression,} \quad (12)$$

$$\epsilon_t = \frac{\Delta d}{d} = \frac{-p}{E} \left[\frac{d^2+c^2}{d^2-c^2} - \nu - \nu \frac{d^2}{d^2-c^2} \right], \quad (13)$$

or

$$\Delta d = -p \frac{d}{E} \left[\frac{d^2(1-\nu)+c^2}{d^2-c^2} - \nu \right]. \quad (14)$$

Define an effective spring rate, K_1 such that

$$2\pi(a+\Delta a) L_1 P_1 = K_1 \Delta a \quad (15)$$

where L_1 is the length of the cylinder on which the internal pressure, P_1 acts.

$$K_1 = 2\pi L_1 P_1 \left(\frac{a}{\Delta a} + 1 \right),$$

but

$$\frac{a}{\Delta a} = \frac{E}{P_1} \frac{1}{\left[\frac{b^2+(1-\nu)a^2}{b^2-a^2} + \nu \right]}, \quad (16)$$

thus

$$K_1 = \frac{2\pi L_1 E}{\frac{b^2+(1-\nu)a^2}{b^2-a^2} + \nu} + 2\pi L_1 P_1. \quad (17)$$

Similarly, define an effective spring rate, K_2 such that

$$2\pi(d+\Delta d) L_2 P_2 = -K_2 \Delta d \quad (18)$$

where L_2 is the length of the cylinder on which the external pressure, P_2 acts.

$$K_2 = 2\pi L_2 P_2 \left(-\frac{d}{\Delta d} - 1 \right), \quad (19)$$

but

$$-\frac{d}{\Delta d} = \frac{E}{P_2} \frac{1}{\left(\frac{d^2(1-\nu)+c^2}{d^2-c^2} - \nu \right)} ; \quad (20)$$

thus,

$$K_2 = \frac{2\pi L_2 E}{\frac{d^2(1-\nu)+c^2}{d^2-c^2} - \nu} - 2\pi L_2 P_2 . \quad (21)$$

The mass associated with the internal pressure which is being expanded is $\pi(b^2-a^2)L_1\rho_s$ where ρ_s is the density of the material.

One equation of motion becomes

$$\pi(b^2-a^2)L_1\rho_s \frac{\ddot{Y}}{2} = \pi L_1 P_1 Y - \left(\frac{2\pi L_1 E}{\left[\frac{b^2+(1-\nu)a^2}{b^2-a^2} + \nu \right]} + 2\pi L_1 P_1 \right) \frac{Y-2a}{2} \quad (22)$$

where $Y=2(a+\Delta a)$ is the instantaneous value of the inner diameter. Δa and Δd are the radial expansions of the a and d radii.

and $\Delta a = \frac{Y-2a}{2}$, $\ddot{Y} = 2\Delta\ddot{a}$.

Rewriting the equation of motion:

$$(b^2 - a^2)\rho_S \ddot{a} = P_1 2a + P_1 2\Delta a - \frac{E 2\Delta a}{\left[\frac{b^2 + (1-\nu)a^2}{b^2 - a^2} + \nu\right]} - P_1 2\Delta a \quad (23)$$

or

$$(b^2 - a^2)\rho_S \ddot{a} = 2aP - \frac{2\Delta a E}{\left[\frac{b^2 + (1-\nu)a^2}{b^2 - a^2} + \nu\right]} \quad (24)$$

Similarly

$$(d^2 - c^2)\rho_S \ddot{d} = -2dP - \frac{2\Delta d E}{\left[\frac{d^2 + (1-\nu)c^2}{d^2 - c^2} - \nu\right]} \quad (25)$$

Effects of Compressibility

The physical volume (V) available to the fluid at any instant is:

$$V = \frac{\pi}{4} (Y^2 - Z^2) L_4 + A_5 (X - L_1 - L_2 - L_3) + V_0 - V_B \quad (26)$$

X is the distance of piston travel from its rearmost position in the rear buffer (see Variable Buffer Orifice Area). See Symbols of Basic Engineering Values, Dimensions, and Physical Parameters for definitions of symbols. By definition, the bulk modulus, β , is:

$$\beta = - \frac{\Delta P}{\Delta V/V} \quad (27)$$

or

$$\Delta P = - \frac{\Delta V}{V} \beta \quad (28)$$

In differential form with respect to time, t;

$$\frac{dP}{dt} = - \frac{\beta}{V} \frac{dV}{dt} \quad (29)$$

but

$$\frac{dV}{dt} = \frac{\pi L_4}{2} (Y\dot{Y} - Z\dot{Z}) + A_5 \dot{X} \quad (30)$$

Then the basic equation becomes:

$$\dot{P}_2 = -\beta \frac{\frac{\pi L_4}{2} (Y\dot{Y} - Z\dot{Z}) + A_5 \dot{X}}{\frac{\pi L_4}{4} (Y^2 - Z^2) + A_5 (X - L_1 - L_2 - L_3) + V_0 - V_B} \quad (31)$$

$$Y = 2a + 2\Delta a$$

$$Z = 2d + 2\Delta d$$

$$\dot{Y} = 2\Delta \dot{a}$$

$$\dot{Z} = 2\Delta \dot{d}$$

$$*\beta = NP + 132,500$$

$$Y^2 = 4(a + \Delta a)^2$$

$$Z^2 = 4(d + \Delta d)^2$$

The desired equation becomes:

$$*\dot{P}_2 = - (NP_2 + 132,500) \frac{2\pi L_4 [(a + \Delta a) \Delta \dot{a} - (d + \Delta d) \Delta \dot{d}] + A_5 \dot{X}}{\pi L_4 [(a + \Delta a)^2 - (d + \Delta d)^2] + A_5 (X - L_1 - L_2 - L_3) + V_0 - V_B} \quad (32)$$

*This formula is expressed in English units.

This equation applies only to P_2 during normal fire, when the piston is between the buffers. If the buffer volumes are added to the equation, this same equation can be applied to P_2 for all three firing conditions (normal, misfire, and cook-off). These additional volumes can be prefixed with controlling functions J, R, and U, to "switch" these portions on or off as the case may be, depending on the location of the piston.

Therefore, the final equation becomes:

$$\begin{aligned} * \dot{P}_2 = & -(NP_2 + 132,500) \frac{2\pi L_4 \left[(\dot{a} + \Delta \dot{a}) \Delta \dot{a} - (d + \Delta d) \Delta \dot{d} \right]}{\pi L_4 \left[(a + \Delta a)^2 - (d + \Delta d)^2 \right] + A_5 (X - L_1 - L_2 - L_3)} \\ & \frac{+RA_5 \dot{X} - J(A_4 \dot{X} + Q_R) + U(A_4 \dot{X} - Q_F)}{+V_0 - V_B - J(A_3 + A_R)X + U(A_4 + A_F)(X - L_1 - L_2 - L_3)} \cdot \end{aligned} \quad (33)$$

The equation for the pressure in the front buffer pocket, P_1 , becomes:

$$* \dot{P}_1 = U(P_1 N + 132,400) \frac{(A_4 \dot{X} - Q_R)}{(A_4 + A_F)(L_1 + L_2 + L_3 - X)} \quad (34)$$

*These equations are expressed in English units.

The equation for the pressure in the rear buffer pocket, P_3 , becomes:

$$*\dot{P}_3 = -J(P_3N + 132,500) \frac{(A_3X + Q_R)}{(A_3 + A_R)X} \quad (35)$$

(see figure 2) .

$$J = 1 \text{ when } X < L_3$$

$$J = 0 \text{ when } X > L_3$$

$$R = 1 \text{ when } X > L_3$$

$$R = 0 \text{ when } X < L_3$$

$$U = 1 \text{ when } X > L_2 + L_3$$

$$U = 0 \text{ when } X < L_2 + L_3$$

Recoil Equation

The recoil equation is

$$*M_R \ddot{X} = S(2.69 P_3(.5) + 2.97 P_1) + P_3 A_3 - P_1 A_4 - HB(t) - W_R \sin \gamma \quad (36)$$

where M_R and W_R are the mass and weight, respectively, of the recoiling parts. $B(t)$ is the breech force (see Appendix A), and H is a control function that initiates $B(t)$ when $\dot{X}=V_e$. $P_3 A_3$ and $P_1 A_4$ are forces on the piston. $*2.69P_3(.5) + 2.97P_1$ is seal friction (see Appendix B), and S is a control function which changes the direction of the friction force when the piston changes direction. γ is the angle of elevation of the weapon.

*These formulas are expressed in English units.

Flow Equations

The classical flow equations applied to the buffers, become:

$$Q_F = A_F K_F \sqrt{\frac{2g}{\omega} | (P_1 - P_2) |} \quad \text{sign of } P_1 - P_2 = \text{sign of } Q_F \quad (37)$$

$$Q_R = A_R K_R \sqrt{\frac{2g}{\omega} | (P_3 - P_2) |} \quad \text{sign of } P_3 - P_2 = \text{sign of } Q_R \quad (38)$$

where Q is the rate of flow in in^3/sec , A is the orifice area at any instant, K is coefficient of discharge, ω is the specific weight of the fluid, and P is the pressure.

For development of this formula and a discussion of coefficient of discharge see Appendix C.

Variable Buffer Orifice Area

Front Buffer Area A_F

Assuming a straight linear taper (see fig 3):

$$\frac{r_2 - r_3}{L_1} = \frac{r_X - r_3}{L_1 + L_2 + L_3 - X} \quad (39)$$

$$(r_2 - r_3) \left(1 + \frac{L_2}{L_1} + \frac{L_3}{L_1} - \frac{X}{L_1} \right) = r_X - r_3 \quad (40)$$

$$r_2 + \frac{r_2 L_2}{L_1} + \frac{r_2 L_3}{L_1} - \frac{r_2 X}{L_1} - r_3 - \frac{r_3 L_2}{L_1} - \frac{r_3 L_3}{L_1} + \frac{r_3 X}{L_1} + r_3 = r_X \quad (41)$$

$$G_F = r_2 - r_3 \quad (42)$$

$$A_1 = \pi (r_2^2 - r_3^2) \quad (43)$$

$$r_X = G_F \frac{L_2}{L_1} + G_F \frac{L_3}{L_1} - G_F \frac{X}{L_1} + r_2 \quad (44)$$

$$r_X = \frac{G_F}{L_1} (L_2 + L_3 - X) + r_2 \quad (45)$$

$$A_F = \pi(r_X^2 - r_3^2) \quad (46)$$

$$A_F = \pi \left[\left(\frac{G_F}{L_1} (L_2 + L_3 - X) + r_2 \right)^2 - r_3^2 \right] \quad (47)$$

$$A_F = \left[\frac{G_F^2}{L_1^2} (L_2 + L_3 - X)^2 + \frac{2G_F r_2}{L_1} (L_2 + L_3 - X) + r_2^2 - r_3^2 \right] \quad (48)$$

$$A_F = A_1 + \frac{\pi G_F^2}{L_1^2} (L_2 + L_3 - X)^2 + \frac{2\pi G_F r_2}{L_1} (L_2 + L_3 - X) \quad (49)$$

$$A_F = A_1 + \frac{\pi G_F^2}{L_1^2} (L_2 + L_3)^2 + \frac{2\pi G_F r_2}{L_1} (L_2 + L_3) - \left[\frac{\pi G_F^2}{L_1^2} (L_2 + L_3)^2 + \frac{2\pi G_F r_2}{L_1} (L_2 + L_3) \right] X + \frac{\pi G_F^2}{L_1^2} X^2 \quad (50)$$

Rear Buffer Area A_R

Assuming a straight linear taper (see fig 4):

$$\frac{r_1 - r_3}{L_3} = \frac{r_X - r_3}{X} \quad (51)$$

$$r_X = \frac{r_1 X - r_3 X + L_3 r_3}{L_3} \quad (52)$$

$$G_R = r_1 - r_3 \quad (53)$$

$$A_R = \pi(r_X^2 - r_3^2) \quad (54)$$

$$A_R = \pi \left[\left(G_R \frac{X}{L_3} + r_3 \right)^2 - r_3^2 \right] \quad (55)$$

$$A_R = \pi \left[G_R^2 \frac{X^2}{L_3^2} + 2G_R \frac{X}{L_3} r_3 + r_3^2 - r_3^2 \right] \quad (56)$$

$$A_R = \frac{2\pi G_R r_3}{L_3} X + \frac{\pi G_R^2}{L_3^2} X^2 \quad (57)$$

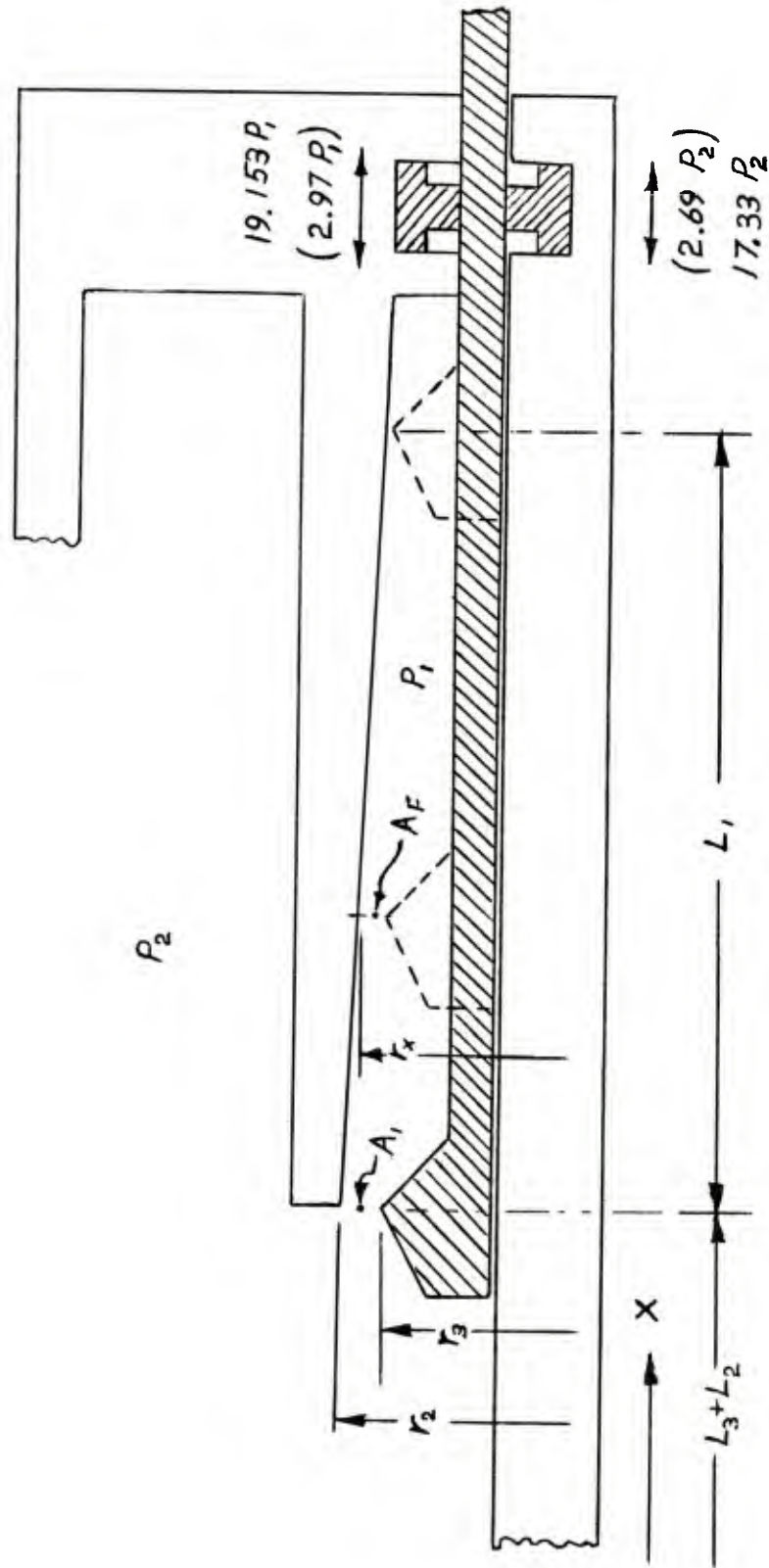


Figure 3. Front buffer.

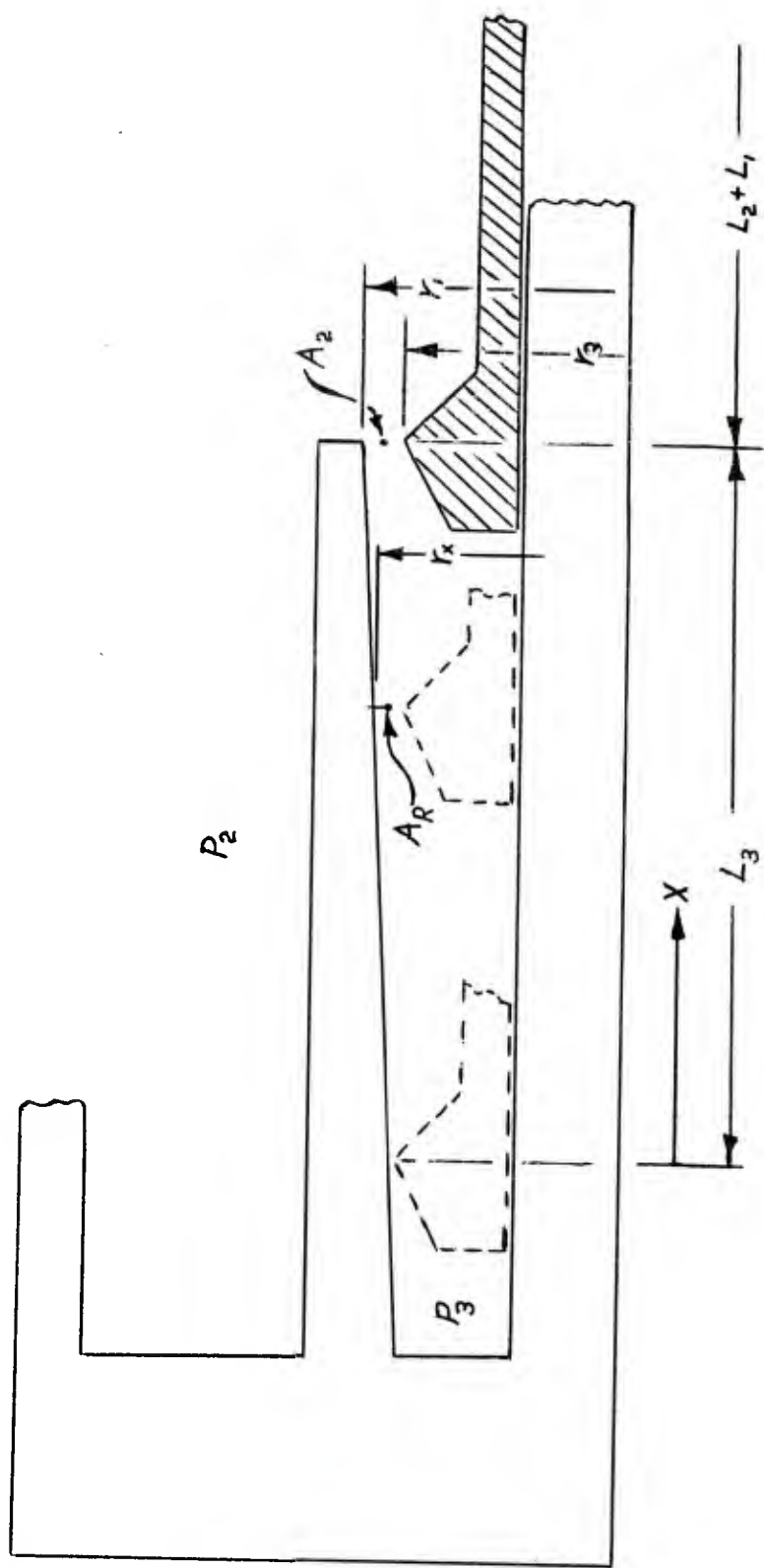


Figure 4. Rear buffer.

Amount of Energy the System is Able to Store

Energy stored in the compressed fluid, E_f , is:

$$E_f = \int P dV \quad (58)$$

where V is the total fluid volume at any instant, and P is the variable pressure.

$$V = \frac{\pi}{4} (Y^2 - Z^2) L_4 + A_5 (X - L_1 - L_2 - L_3) + V_0 - V_B. \quad (59)$$

X is the distance of piston travel from the rearmost position in the rear buffer. A_5 is the effective net piston area. L_4 is the effective length of both inside (Z) and outside (Y) cylinders.

$$\Delta P = - \frac{\Delta V}{V} \beta \quad (\text{eqn 28}) \quad (60)$$

$$\frac{dP}{dt} = - \frac{\beta}{V} \frac{dV}{dt} \quad (61)$$

$$dV = - \frac{V}{\beta} \frac{dP}{dt} dt \quad * \beta = NP_2 + 132,500. \quad (62)$$

Now if these last two equations are substituted into the energy equation one has:

$$*E_f = \int P_2 - \frac{V}{(NP_2 + 132,500)} \frac{dP}{dt} dt \quad (63)$$

or

$$*E_f = - \int \frac{VP_2 \dot{P}_2}{NP_2 + 132,500} dt \quad (64)$$

Energy stored in the flexible cylinder walls, E_c , is:

$$E_c = \int 2(a + \Delta a) \pi L_4 P d(\Delta a) + \int 2(d + \Delta d) \pi L_4 P d(\Delta d) \quad (65)$$

$$Y = 2(a + \Delta a)$$

$$Z = 2(d + \Delta d)$$

$$\dot{Y} = 2\Delta \dot{a}$$

$$\dot{Z} = 2\Delta \dot{d}$$

*These formulas are expressed in English units.

$$E_C = \int Y \pi L_4 P \frac{d(\Delta a)}{dt} dt + \int Z \pi L_4 P \frac{d(\Delta d)}{dt} dt , \quad (66)$$

$$E_C = \pi L_4 \int Y P \Delta \dot{a} dt + \pi L_4 \int Z P \Delta \dot{d} dt , \quad (67)$$

and finally,

$$E_C = \frac{\pi L_4}{2} \int Y \dot{Y} P_2 dt + \frac{\pi L_4}{2} \int Z \dot{Z} F_2 dt , \quad (68)$$

or

$$E_C = 2\pi L_4 \int (a + \Delta a) \Delta \dot{a} P_2 dt + 2\pi L_4 \int (d + \Delta d) \Delta \dot{d} P_2 dt . \quad (69)$$

Summary of Mathematical Model

A summary of the model, using all the physical phenomena taking place in the recoil mechanism, may be expressed by the following ten equations. The equations include the ten unknown variables defined in the next section.

$$Q_F = A_F K \sqrt{\frac{2g}{\omega} |P_1 - P_2|} \quad \text{sign of } (P_1 - P_2) = \text{sign of } Q_F \quad (70)$$

$$Q_R = A_R K \sqrt{\frac{2g}{\omega} |P_3 - P_2|} \quad \text{sign of } (P_3 - P_2) = \text{sign of } Q_R \quad (71)$$

$$A_F = A_1 + \frac{\pi G_F^2}{L_1^2} (L_2 + L_3)^2 + \frac{2\pi G_F r_2}{L_1} (L_2 + L_3) - \left[\frac{\pi G_F^2 2}{L_1^2} (L_2 + L_3) + \frac{2\pi G_F r_2}{L_1} \right] X + \frac{\pi G_F^2}{L_1^2} X^2 \quad (72)$$

$$A_R = \frac{2\pi G_R r_3}{L_3} X + \frac{\pi G_R^2}{L_3^2} X^2 \quad (73)$$

$$(b^2 - a^2) \rho_S \ddot{\Delta a} = 2aP_2 - \frac{2E\Delta a}{\left[\frac{b^2 + (1 - \nu_S)a^2}{b^2 - a^2} + \nu_S \right]} \quad (74)$$

$$(d^2 - c^2) \rho_S \ddot{\Delta d} = -2dP_2 - \frac{2E\Delta d}{\left[\frac{d^2 + (1 - \nu_S)c^2}{d^2 - c^2} - \nu_S \right]} \quad (75)$$

$$*\dot{P}_1 = U(P_1N + 132,500) \frac{(A_4 \dot{X} - Q_R)}{(A_4 + A_F)(L_1 + L_2 + L_3 - X)} \quad (76)$$

$$*\dot{P}_2 = -(NP_2 + 132,500) \frac{2\pi L_4 \left[(a + \Delta a) \dot{\Delta a} - (d + \Delta d) \dot{\Delta d} \right]}{\pi L_4 \left[(a + \Delta a)^2 - (d + \Delta d)^2 \right] + A_5 (X - L_1 - L_2 - L_3)} \\ + \frac{+RA_5 \dot{X} - J(A_4 \dot{X} + Q_R) + U(A_4 \dot{X} - Q_F)}{+V_0 - V_B - J(A_3 + A_R)X + U(A_4 + A_F)(X - L_1 - L_2 - L_3)} \quad (77)$$

$$*\dot{P}_3 = -J(P_3N + 132,500) \frac{(A_3 X + Q_R)}{(A_4 + A_R)X} \quad (78)$$

$$J = 1 \text{ when } X < L_3 \\ J = 0 \text{ when } X > L_3$$

$$R = 1 \text{ when } X > L_3 \\ R = 0 \text{ when } X < L_3$$

$$U = 1 \text{ when } X > L_2 + L_3 \\ U = 0 \text{ when } X < L_2 + L_3$$

$$*M_R \ddot{X} = S(2.69(.5)P_3 + 2.97P_1) + P_3A_3 - P_1A_4 - HB(t) - W_R \sin \gamma \quad (79)$$

$$S = -1 \text{ when } \dot{X} > 0 \\ S = +1 \text{ when } \dot{X} < 0$$

$$H = 1 \text{ when } \dot{X} > V_e \\ H = 0 \text{ when } \dot{X} < V_e$$

Ten Unknown Variables

The relationships of the following ten unknown variables are expressed by the mathematical model. Fixing their relationships will unlock other relationships and parameters.

*These equations are expressed in English units.

- Q_F = Fluid flow through annular orifice in front buffer
 Q_R = Fluid flow through annular orifice in rear buffer
 P_1 = Pressure in front buffer area
 P_2 = Pressure in recoil cylinder
 P_3 = Pressure in rear buffer area
 A_F = Variable orifice area in front buffer
 A_R = Variable orifice area in rear buffer
 X = Distance of piston travel from its innermost position in rear buffer
 Δa = Radial expansion of inside radius of outside recoil cylinder
 Δd = Radial expansion of outside radius of inside sleeve

Other Relationships of Important Parametric Design Values

$$A_1 = \pi(r_2^2 - r_3^2) \quad (80)$$

$$A_2 = \pi(r_1^2 - r_3^2) \quad (81)$$

$$A_3 = \pi(r_3^2 - r_5^2) \quad (82)$$

$$A_4 = \pi(r_3^2 - r_4^2) \quad (83)$$

$$A_5 = \pi(r_4^2 - r_5^2) = A_3 - A_4 \quad (84)$$

$$G_F = r_2 - r_3 \quad (85)$$

$$G_R = r_1 - r_3 \quad (86)$$

$$\sigma_t = \frac{(b^2 + a^2)}{(b^2 - a^2)} P_2 \quad (\text{see ref 2}) \quad (87)$$

$$\sigma_a = \frac{a^2}{b^2 - a^2} P_2 \quad (88)$$

$$\sigma_r = -P_2 \quad (89)$$

$$2\sigma_e^2 = (\sigma_t - \sigma_r)^2 + (\sigma_r - \sigma_a)^2 + (\sigma_a - \sigma_t)^2 \quad (90)$$

$$V = \pi(a^2 - d^2)L_4 + A_5[X_0 - (L_1 + L_2 + L_3)] + V_0 - V_B \quad (91)$$

$$V_B = (L_4 - L_2) (R_{av}^2 - r_{ab}^2) \pi \quad (92)$$

$$W_f = V_1 \omega \quad (93)$$

$$*E_f = - \int \frac{VP_2 \dot{P}_2}{NP_2 + 132,500} dt \quad (94)$$

$$E_C = \frac{\pi L_4}{2} \int Y \dot{P}_2 dt + \frac{\pi L_4}{2} \int Z \dot{P}_2 dt \quad (95)$$

$$*X_0: \quad p = \frac{\pi L_4 [(2a + \Delta a) \Delta a + (2d - \Delta d) \Delta d] + A_5(X_0 - X)}{\pi L_4 (a^2 - d^2) + A_5[X_0 - (L_1 + L_2 + L_3)] - V_B + V_0} (NP - 132,500) \quad (96)$$

$$C = \frac{100}{\pi L_4 (a^2 - d^2) + A_5[X_0 - (L_1 + L_2 + L_3)] - V_B + V_0} \quad (97)$$

$$P_{cr} = \frac{\pi^2 EI}{4(L_1 + L_2 + L_3 - X_{min})^2} \quad (\text{Euler's formula}) \quad (98)$$

$$I = \frac{\pi(r_4^4 - r_3^4)}{4} \quad (99)$$

$$P_{pt} = A_5 P_2 \quad (100)$$

or

$$= A_3 P_3 - A_4 P_2 \quad (101)$$

*These equations are expressed in English units.

**Symbols of Basic Engineering Values, Dimensions
and Physical Parameters**

L_1	= run-up distance in front buffer
L_2	= run-up distance for normal firing (between buffers)
L_3	= run-up distance in rear buffer
L_4	= length of outside flexible cylinder
r_1	= inside radius of rear buffer at its entrance
r_2	= inside radius of front buffer at its entrance
r_3	= outside radius of piston over orifice cam
r_4	= outside radius of piston sleeve
r_5	= inside radius of piston sleeve
a	= inside radius of outside flexible cylinder
b	= outside radius of outside flexible cylinder
c	= inside radius of sleeve
d	= outside radius of sleeve
V_B	= approximate volume of fluid the buffers displace
V_0	= extra fluid volume in associated container (if necessary)
V	= total fluid volume in recoil cylinder
W_f	= total weight of fluid

V_e	= firing velocity of recoil mechanism or maximum run-up velocity
V_l	= piston velocity (going into latch)
V_m	= minimum piston velocity (misfire only)
$P_1 \text{ max}$	= maximum pressure in front buffer area
$P_2 \text{ max}$	= maximum pressure in recoil cylinder
$P_3 \text{ max}$	= maximum pressure in rear buffer area
$P \text{ min}$	= minimum pressure in any of the volumes
$P \text{ init}$	= initial pressure in all volumes when piston is at a point of latch
X_0	= the value of X calculated for the point where all pressures are zero
X_{p_0}	= the value of X at the point where $P_2 = 0$ (taken from computer data)
$X \text{ max}$	= maximum value of X in front buffer
$X \text{ min}$	= minimum value of X in rear buffer
E_f	= amount of energy absorbed in the fluid
E_c	= amount of energy absorbed in the cylinder walls
E_{br}	= impulse energy at breech
Imp	= maximum firing impulse at breech
σ_t	= tangential stress in recoil cylinder walls
σ_a	= axial stress in recoil cylinder walls
σ_r	= radial stress in recoil cylinder walls
σ_e	= equivalent stress in recoil cylinder walls

A_1	= annular cross-sectional area between front buffer entrance and piston cam
A_2	= annular cross-sectional area between rear buffer entrance and piston cam
A_3	= annular cross-sectional area of piston through cam
A_4	= difference between areas A_3 & A_5
A_5	= annular cross-sectional area of piston sleeve through shank
A_{S_1}	= side pressure area of inside seal
A_{S_2}	= side pressure area of outside seal
G_F	= radial gap of front buffer at entrance
G_R	= radial gap of rear buffer at entrance
P_{cr}	= critical axial buckling load on end of piston
P_{pt}	= actual maximum load on end of piston
I	= moment of inertia of piston in bending
C	= compression of fluid at latch in %
R_{av}	= average outside radius of both buffers
r_{av}	= average inside radius of both buffers

Fixed, Unchangeable Constants

g	$= 9.81 \text{ m/sec}^2 \text{ (386.088 in/sec}^2\text{)}$	acceleration of gravity
E	$= 199.95 \times 10^9 \text{ Pa (29} \times 10^6 \text{ psi)}$	modulus of elasticity for steel
ν_s	$= 0.287$	Poisson's ratio for steel
ρ_s	$= 798.685 \text{ kg sec}^2/\text{m}^4 \text{ (.000,732,994 lb sec}^2/\text{in.}^4\text{)}$	density of steel
ω	$= 939.5 \text{ kg/m}^3 \text{ (.033,942 lb/in.}^3\text{)}$	specific weight of Dow Corning 200 silicon fluid at atmospheric pressure
ν_R	$= .48$	Poisson's ratio for Nitrile rubber
μ_R	$= .50$	dynamic coefficient of friction for rubber
M_R	$= 300.65 \text{ kg sec}^2/\text{m (16.8355 lb sec}^2/\text{in.)}$	mass of recoiling parts
W_R	$= 2948.4 \text{ kg (6500 lb)}$	weight of recoiling parts
$B_{(t)}$	$= \text{(see appendix A)}$	firing impulse at breech (variables)
N	$= 12.67$	tangent of slope of bulk modulus curve
β	$= NP+9,1356 \text{ (NP+132,500)}$	bulk modulus (see appendix D)
K	$= .95$	flow coefficient (see appendix C)
γ	$= 75^\circ, 45^\circ, 10^\circ$	angle of elevation of weapon
$H, J, R, S \& U$		control functions (see page 18)
$^*\sigma_{\text{max}}$	$= 5.516 \times 10^8 \text{ Pa (80,000 psi)}$	maximum allowable tensile stress in all cylinder parts (safety factor approximately 2)

*Assuming steel used is

AISI 4130 or 4140 QQ-S-624 or equal.

Rc 36-40 ultimate tensile stress - $1.2411 \times 10^9 \text{ Pa (180,000 psi)}$

Tensile yield stress - $1.1238 \times 10^8 \text{ Pa (163,000 psi)}$

COMMENTS PERTAINING TO THE MATHEMATICAL MODELS

Several mathematical models were developed before the final one shown on the preceding pages was derived. Many computer studies were programmed and run based upon these models. Certain important facts emerged from these studies.

Professor A. P. Boresi of the University of Illinois developed a finite elements model and 26 computer runs were made with this program. The results did not solve all of the problems, since this was only a static model. What was established, however, was how thin the cylinder walls could be without being overstressed. In addition, the magnitude of wall deflection and the approximate amount of energy which the walls would be capable of absorbing were determined.

Since the problem is actually a dynamic problem, another computer program was set up with all the velocities and masses taken into account. Mr. Joe Wilson ran this program on an analog computer. A total of 27 runs were made. Since the analog computer is limited in the quantity of unknowns it can solve for, this study was also restricted in its scope. However, it did establish the run-up velocity and the run-up distance required to achieve that velocity.

The factor which will ultimately determine how well the system works, however, is not the amount of energy stored or the run-up velocity, but if the recoiling mass consistently, but just barely returns to the latch position (with all the breech energy spent). Of course, equally important is the piston force (rod pull) transmitted to the structure. The longer the run-up-distance, the smaller this force may be and the softer the recoil will be. These facts show up clearly in the analog printouts.

Finally, after several attempts, one all-inclusive mathematical model was written which was valid for all three firing conditions (normal, mis-fire, and cook-off) and for any possible position of the piston. The development of this model is summarized in equations 70 through 79. With this model the overwhelming amount of input and output data had to be limited so as not to obscure the most important relationships. However, a tool was now available which could be used for a trial-and-error type of optimization of the most important parameters. About 100 computer runs were made. Of these, eight were good runs which are summarized in the following paragraph. (Mr. Philip Benzkofer programmed and ran this study on a digital computer.)

RESULTS OF THE COMPUTER RUNS

Table 1 shows the data which were held constant during the 8 "good" computer runs. One of these runs is shown as appendix E.

Figure 5 is a to-scale drawing based upon the dimensions given in table 1. The drawing is merely a concept study in that no thought was given as to how the mechanism could be broken up into smaller parts for ease of machining and assembly.

Table 2 lists relevant engineering information calculated from some of the computer data.

Table 3 has pressures, stresses, and other optimized values from the computer runs.

DISCUSSION

The thought of a compressible liquid recoil system opens up the possibility of a greatly improved and simplified recoil mechanism. There would appear to be a likelihood of storing a large amount of energy in a small space, thus achieving a greatly simplified and "clean" one-principle, one-fluid design. There would be fewer tanks and fewer sub-mechanisms, fewer parts overall. Therefore, the main criterion that has been adhered to throughout the program is simplification.

The RESULTS give a set of ideal dimensions and parameters for the CFSR mechanism which are discussed in detail in the following paragraph.

Recoil Cylinder (Fig 5)

The outside diameter of the recoil cylinder is 60.20 cm (23.7 in.). The cylinder surrounds (is coaxial with) the 33.02 cm (13-inch) diameter barrel (cannon tube). The effective length of the cylinder is 215.9 cm (85 in.). There remains a rather small space for the fluid. This small cylinder size is made possible only by allowing different parameters for the 3 different firing conditions, i.e., the cylinder still does not hold enough fluid to function properly during misfire. Recognizing how rarely a misfire occurs, this gives the designer a choice of whether or not to design for misfire. The only thing that probably would occur with the system underdesigned for misfire is that the piston would draw a vacuum which would cause cavitation and foaming. This may not be serious if it does not occur too frequently.

Table 1 Computer run constants

Parameter	Value mm (in.)
Run-up distance in front buffer (L_1)	508 mm (20 in.)
Run-up distance for normal firing (between buffers) (L_2)	508 mm (20 in.)
Run-up distance in rear buffer (L_3)	508 mm (20 in.)
Length of outside flexible cylinder (L_4)	2159 mm (85 in.)
Inside radius of rear buffer, at its entrance (r_1)	208.3 mm (8.20 in.)
Inside radius of front buffer at its entrance (r_2)	208.3 mm (8.20 in.)
Outside radius of piston over orifice cam (r_3)	207 mm (8.15 in.)
Outside radius of piston sleeve (r_4)	193.7 mm (7.625 in.)
Inside radius of piston sleeve (r_5)	184.2 mm (7.25 in.)
Inside radius of outside flexible cylinder (a)	285.8 mm (11.25 in.)
Outside radius of outside flexible cylinder (b)	301 mm (11.85 in.)
Inside radius of sleeve (c)	165.1 mm (6.50 in.)
Outside radius of sleeve (d)	184.2 mm (7.25 in.)
Outer wall thickness (b-a)	15.2 mm (.60 in.)
Inner wall thickness (d-c)	19.1 mm (.75 in.)
Front buffer gap (G_F)	1.3 mm (.05 in.)
Rear buffer gap (G_R)	1.3 mm (.05 in.)
Annular effective front buffer gap area (A_1)	65.2 mm ² (2.568 in. ²)
Annular effective rear buffer gap area (A_2)	65.2 mm ² (2.568 in. ²)

Table 1 (Cont'd)

Parameter	Value mm (in.)
Piston sleeve thickness ($r_4 - r_5$)	9.5 mm (0.375 in.)
Annular effective cross sectional piston area (A_5)	445.1 mm ² (17.524 in. ²)
Moment of inertia for cross section of piston (I)	20,187 cm ⁴ (485 in. ⁴)
Maximum critical buckling load on end of piston (P_0)	5,203,792 kg (11,472,396 lb)
Initial pressure at latch (P_{init})	20,684.2 kPa (3000 psi)
Average outside radius of both buffers (R_{av})	232.4 mm (9.15 in.)
Average inside radius of both buffers (r_{av})	207 mm (8.15 in.)
Approximate volume of buffers, filling oil space (V_p)	34,113 cm ³ (2,081.7 in. ³)
Impulse energy at breech (E_{br})	597,496 Nn (5,288,287 in.-lb)

Table 2. Calculated data

Extra fluid volume in connected container (V_o)	0	.164 m ³ (10,000 in. ³) / 164 litre (43.29 gal)	0	.164 m ³ (10,000 in. ³) / 164 litre (43.29 gal)
Approximate volume of the two buffers (V_B)	0	0	.0339 m ³ (2,081 in. ³)	.0339 m ³ (2,081 in. ³)
Total fluid volume in recoil cylinder (V)	3186 m ³ (19,543 in. ³) / 318.6 litre (84.60 gal)	.485 m ³ (29,724 in. ³) / 485 litre (128.67 gal)	.2599 m ³ (15,948 in. ³) / 259.9 litre (69.04 gal)	.4259 m ³ (26,126 in. ³) / 425.9 litre (113.10 gal)
Total weight of fluid (W_f)	300.9 kg (663.3 lb)	457.6 kg (1,008.9 lb)	245.5 kg (541.3 lb)	402.2 kg (886.8 lb)
*The value of X calculated for the point where all pressures are zero (X_o)	1209 mm (47.6 in.)	1470.7 mm (57.9 in.)	1117.6 mm (44.0 in.)	1376.7 mm (54.2 in.)
Compression of fluid at latch in % (C)	1.7572 %	1.7624 %	1.7579 %	1.7570 %
C at smallest X	2.6464 %	2.3471 %	2.8476 %	2.4222 %

*X = distance of piston travel from its innermost position in rear buffer.

Table 3. Basic engineering values calculated from computer runs; $V_B = 0^*$

Run no. Mode of fire	1	2	3	4	5	6	7	8
		$H = 0 \text{ when } \dot{X} < V_e$		$H = 1 \text{ when } \dot{X} \geq V_e$		$H = 0$ Misfire		$H = 1$ Cook-off
γ^a	0	0	0	45	75	0	0	0
V_e	0	0	0	0	0	0	163.871 (10,000)	0
$P_1 \text{ max}$	2.863x10 ⁷ (4,152)	2.881x10 ⁷ (4,179)	2.847x10 ⁷ (4,129)	2.935x10 ⁷ (4,258)	2.959x10 ⁷ (4,281)	2.068x10 ⁷ (2,999)	2.418x10 ⁷ (3,507)	3.147x10 ⁷ (4,584)
$P_2 \text{ max}$	2.863x10 ⁷ (4,152)	2.881x10 ⁷ (4,179)	2.847x10 ⁷ (4,129)	2.935x10 ⁷ (4,258)	2.959x10 ⁷ (4,281)	2.068x10 ⁷ (2,999)	2.418x10 ⁷ (3,507)	3.147x10 ⁷ (4,584)
$P_3 \text{ max}$	3.833x10 ⁷ (5,560)	4.024x10 ⁷ (5,837)	3.840x10 ⁷ (5,559)	4.273x10 ⁷ (6,197)	4.444x10 ⁷ (6,445)	2.068x10 ⁷ (2,999)	2.068x10 ⁷ (3,000)	6.936x10 ⁷ (10,060)
$P \text{ min}$	-1.248x10 ⁶ (-181)	9.255x10 ⁶ (1,342)	8.714x10 ⁶ (974)	1.303x10 ⁶ (-189)	2.241x10 ⁶ (-325)	-5.792x10 ⁶ (-840)	3.034x10 ⁶ (44)	1.882x10 ⁷ (2,439)
σ_t	5.514x10 ⁸ (79,977)	5.550x10 ⁸ (80,497)	5.485x10 ⁸ (79,546)	5.656x10 ⁸ (82,027)	5.699x10 ⁸ (82,654)	3.984x10 ⁸ (57,778)	3.984x10 ⁸ (57,781)	6.061x10 ⁸ (87,912)
σ_a	2.614x10 ⁸ (37,912)	2.631x10 ⁸ (38,159)	2.600x10 ⁸ (37,708)	2.681x10 ⁸ (38,884)	2.702x10 ⁸ (39,182)	1.888x10 ⁸ (27,381)	1.888x10 ⁸ (27,381)	2.873x10 ⁸ (41,874)
σ_r	-2.863x10 ⁸ (-4,152)	-2.881x10 ⁸ (-4,179)	-2.847x10 ⁸ (-4,129)	-2.935x10 ⁸ (-4,258)	-2.959x10 ⁸ (-4,281)	-2.068x10 ⁸ (-2,999)	-2.068x10 ⁸ (-3,000)	-3.147x10 ⁸ (-4,584)
σ_e	3.552x10 ⁸ (51,518)	3.575x10 ⁸ (51,853)	3.533x10 ⁸ (51,241)	3.643x10 ⁸ (52,839)	3.670x10 ⁸ (53,243)	2.568x10 ⁸ (37,219)	2.568x10 ⁸ (37,220)	3.905x10 ⁸ (56,830)
$\Delta a \text{ max}$.693 (.0273)	.856 (.0274)	.688 (.0271)	.711 (.0280)	.715 (.0282)	.591 (.0197)	.500 (.0197)	.762 (.030)
$\Delta d \text{ max}$	-1.196 (-.0077)	-1.198 (-.0078)	-1.196 (-.0077)	-1.201 (-.0079)	-1.203 (-.0080)	-1.142 (-.0056)	-1.142 (-.0056)	.216 (.0085)
E_{br}	8.575 (75,896)	1.140(10,092)	985.1 (8,719)	8863.8 (78,452)	7,983.7 (70,662)	67,232.5 (595,058)	—	3,394 (30,043)
E_f	174.7 (1,546)	306.5 (2,713)	259.2 (2,294)	39.4 (349)	137.4 (1,216)	15,688.4 (136,854)	—	887.4 (7,854)
E_c	—	—	—	—	—	—	—	—
P	70,693 (155,852)	75,751 (167,002)	70,708 (155,880)	81,987 (180,751)	88,672 (191,080)	23,893 (52,542)	23,841 (52,560)	153,478 (338,356)
P_{or}	—	—	—	—	—	—	—	—
P_t	—	—	—	—	—	—	—	—
X	111.76 (44)	—	—	—	—	—	—	—
X_{po}	114.3 (45)	88.9 (35)	99.1 (39)	110.5 (43.5)	108.7 (42.8)	124.5 (49)	147.1 (57.9)	—
X_{max}	25.4 (10)	24.1 (9.5)	25.1 (9.8)	23.1 (9.1)	22.4 (8.8)	149.8 (59)	152.4 (60)	—
X_{min}	—	—	—	—	—	—	—	—
V_e cm/sec (in./sec)	640 (252)	566.4 (223)	607.1 (239)	571.5 (225)	546 (215)	624.8 (246)	678.2 (267)	—
V_f cm/sec (in./sec)	462.3 (182)	467.4 (184)	464.8 (183)	447 (176)	444.5 (175)	—	—	—
V_m cm/sec (in./sec)	—	—	—	—	—	1.85 (.73)	1.75 (.69)	—

*For definitions of symbols in left-hand column see paragraphs 8, 10, or 11 as appropriate.

Another alternative is to add $.1639 \text{ m}^3$ ($10,000 \text{ in.}^3$) of fluid in a container connected to the recoil cylinder. As shown in tables 2 and 3, this would maintain positive pressures throughout the misfire cycle. However, the diameter of the recoil cylinder cannot be increased to accommodate the extra fluid without also increasing the wall thickness, which would change wall deflection, energy absorbing qualities, total weight, and probably most of the other parameters as well.

The maximum stresses in the cylinder wall are close to the pre-set limit of $5.516 \times 10^8 \text{ Pa}$ ($80,000 \text{ psi}$), which gives a static safety factor of about $2\frac{1}{2}$. The actual maximum stress is a tangential stress equal to $6.0613 \times 10^8 \text{ Pa}$ ($87,912 \text{ psi}$) and is acceptable. This assumes steel such as AISI 4130 or 4140, or their equivalent, is used throughout.

It was desired to keep the recoil cylinder long and narrow. This would give a low weapon profile and a "clean" efficient design. Since a cylinder longer than 2.44 or 2.74 meters (8 or 9 feet) is very difficult to machine, the effective length was set at 2.13 to 2.44 meters (7 to 8 feet).

The cylinder is 15.2 mm (0.60 inch) thick and is intended to deflect approximately .50 mm (0.0197 inch) (piston at latch) to absorb some of the energy.

The inner cylinder (sleeve) over which the piston slides is heavier (19.1 mm (0.75 inch) thick) to minimize deflection .142 mm (0.0056 inch) at latch). If it deflects too much, it could cling to the barrel which slides inside the sleeve and prevent the mechanism from functioning properly.

At one time, a multi-cylinder recoil system was considered. This would be a system with several recoil cylinders in a cluster around the barrel. After considering all the pros and cons, however, the idea of a multi-cylinder recoil system was discarded, mainly for the sake of simplicity and reliability.

Dow Corning 200 Silicone Fluid

The volume and weight of the silicone fluid was finally reduced to $.322 \text{ m}^3$ (85 gallons) 300.73 kg/663 pounds excluding the extra volume $.1639 \text{ m}^3$ (10,000 in.³) which, if included, would total $.488 \text{ m}^3$ (129 gallons), 457.67 kg (1,009 pounds). The volume of fluid displaced by the buffers $.0339 \text{ m}^3$ (2,081 in.³) was not taken into account in the computer program since it was very uncertain initially what size and shape they would take. It would be advisable to design the buffers so that they occupy as little space as possible but still remain stiff. However, to compensate for the buffer volumes, which originally were assumed to be all fluid, an equivalent amount of fluid must be added to the cylinder volume. One way of doing this, would be to make the recoil cylinder 25.4 cm (10 inches) longer. This is shown in figure 5 as a 12.7 cm (5-inch) extension at each end of the 215.9 cm (85-inch) working length of the cylinder.

The silicon fluid is compressed only about 1.76% at latch during normal fire. The initial pressure at latch is $2.068 \times 10^7 \text{ Pa}$ (3,000 psi) ($P_1 = P_2 = P_3$). During cookoff, the compression reaches 2.85%. This is very low considering the compression could be 3 to 4% before the bulk modulus changes noticeably. Very high pressures in the recoil cylinder, however, would invalidate the linear relationship used in the mathematical model for the bulk modulus and make all the results erroneous.

The values used for the bulk modulus of the silicone fluid are all taken at room temperature. Since the bulk modulus is sensitive to temperature variations, this system would probably not work well in extreme climatic conditions without some modifications.

It has been suggested the effective bulk modulus be made softer or variable by incorporating a gas bag or spring-loaded expansion piston inside the recoil cylinder. The author, however, is opposed to adding anything to complicate the system unless it is absolutely necessary. (The system apparently works well in its present form.)

Energy Stored

The computer was programmed to integrate the energies stored in the cylinder walls (E_c), and in the fluid (E_f), by a method of incremental summation. The values, however, which are listed in table 3, vary considerably from run to run and appear to be totally unreliable. Some of the values are only 1 to 2% of the breech energy. The only thing which these values clearly show is that E_f is many times larger than E_c . It is not the 50-50 relationship which was assumed at the beginning of the program. It would appear that the walls could be designed to store a larger percentage of the energy.

Knowledge of the actual amount of energy which is stored in the system is not essential, however. The only criterion for the feasibility of the system is that the recoiling mass is pushed back, after run-up, by the breech impulse to just beyond the latch position so there is positive latching every time. This is accomplished with the present design.

Piston (Fig 5)

The piston is a long, thick cylinder (sleeve) with a 38.83 cm (14.5 inch) inside diameter and 9.53 mm (0.375 inch) thick walls. Its design is also the result of compromises. The ideal piston would be long and fairly thin, which would accommodate a long soft stroke with low pressures. However, with the piston too thin, its walls would crumble under the working pressures. Present wall thickness 9.53 mm (0.375 inch) is the minimum advised.

Calculations show that the critical buckling load of 5,203,792 kg (11,472,396 pounds) is far greater than the actual maximum load. The critical buckling load is questionable, however, since this is a very difficult value to determine at present (ref 3).

As a result of the piston being fairly thick, its working stroke is necessarily short. The maximum stroke is 64.26 cm (25.3 inch) for normal fire. The piston force, therefore, becomes fairly large. The maximum pressure occurring during cook-off surges to 6.936×10^7 Pa (10,060 psi) which results in a rod pull of 153,476 kg (338,356 lb).

The average maximum piston load for normal fire is 77,111 kg (170,000 lb). These high loads are the result of trying to keep the weight of the oil down, and of a fairly thick-walled piston. However, the peak loads last only a fraction of a second.

The outer G-T seal "sees" a pressure of about 3.172×10^7 Pa (4,600 psi) which is acceptable for this type of seal. (See appendix B.) The P_3 , however, surges to 6.936×10^7 Pa (10,060 psi) and the inside seal "sees" some of this pressure. This peak pressure, as mentioned earlier, is of a very short duration and only reaches the inside seal through a long, tight squeeze under the piston and is consequently not considered critical.

The size and shape of the piston head is unimportant, except that it should have a sharp edge as shown in figure 5. This edge is part of the orifice and its sharpness eliminates much of the friction in the orifice, creating a more predictable flow coefficient. (See appendix C.)

Because of the relatively high initial pressures in the recoil cylinder and the relatively large effective piston area, the latch velocity is fairly high. It varied from 444.5 cm/sec (175 in./sec) to 482.6 cm/sec (190 in./sec). It appeared to be impossible to make this value any smaller without sacrificing some other elements of the design. Consequently, the latch mechanism must be capable of absorbing very large forces.

The firing or run-up velocities for normal fire varied from 546.1 cm/sec (215 in./sec) to 640 cm/sec (252 in./sec) in a run-up distance of 38.1 to 63.5 cm (15 to 25 in.). (This is at point of fire, after ignition delay or "coast" period.) There is, however, no real coast period since the pressure is always present and the velocity continues to increase. The fire control mechanism must start the firing operation at a lower velocity in anticipation of the desired firing velocity being reached. This will probably cause a problem since various ammunition has different time delays.

The run-up velocity, of course, must be adjustable for different zones of fire and different weapon elevations. Computer runs were made for 0° , 45° , and 75° elevations. The pressures, stresses, and velocities seemed to increase very little for the higher elevations and are very acceptable.

Buffers (Fig 5)

The two buffers are identical, each having 50.8 cm (20 in.) effective buffing distance. The size of the buffer gap (annular orifice) was given much attention. It was recognized that the smaller this gap was at the beginning of buffing, the more effective the buffing would be in slowing down the mass. With a small gap, however, the tolerances and machining become very critical. This concerns gaps in the range of .254 mm (0.01 in.). Consequently, a gap of 1.27 mm (0.05 in.) which appeared satisfactory, was selected. Both buffers taper down from a gap of 1.27 mm (0.05 in.) to zero in 50.8 mm (20 in.). The outside of the buffers are also tapered. This, however, is for strength and rigidity. It is realized that a "give" of .254 mm (0.01 in.) in the buffers would be too much, and would disrupt the closely programmed buffing cycle.

There is one condition, however, when the front buffer seems inadequate. This is during misfire. It appeared that no matter how long the buffer was made, or how steep the taper, the piston would slow down but never quite stop before it would hit the bottom of the buffer pocket. To eliminate this condition, there was a suggestion to incorporate a stiff helical spring outside the front buffer. A 35.56 cm (14.) diameter by 15.24 cm (6 in.) long spring, which would compress about 3.81 cm (1.5 in.) was proposed. The author again is opposed to adding any kind of extra mechanism to disrupt the otherwise simple and "clean" design. As an alternative, the use of a "liquid spring" is suggested.

The elements are already present to make a liquid spring (see fig 5). All that is required is to make a straight bore at the end of the taper in the front buffer, and to shape the piston with a "flat" cylindrical surface to fit fairly tight into the bore like a typical piston. One would thus have a liquid spring.

Another solution would be to put an elastic pad (fig 5) inside the bottom of the front buffer as a stopper for the piston. This also seems feasible considering the velocity of the piston at this point is very low about 17.8 mm/sec (0.7 in./sec). A bumper pad of this type should probably be added in any case.

Summary

In conclusion, the "undesirable" and the "desirable" features of the present, CFSR system will be repeated.

The undesirable points are:

1. Very high latch velocity.
2. Very high piston forces.
3. Piston hits bottom of front buffer.
4. Cavitation during some cycles.
5. Fairly large liquid volume and weight.

One feature which may be a disadvantage is that the recoil cylinder can be clamped only at its ends. The rest of the cylinder must be allowed to expand freely. Another feature which may be a disadvantage is the need to be able to "cock" the weapon in the field after a misfire. The piston must be pushed back to the latch position and the pressure raised to 2.068×10^7 Pa (3,000 psi). As had been pointed out in the discussion, however, it is possible to design around most of these undesirable features.

The desirable features are (1) simple and reliable design with few parts, and (2) less maintenance required on the finished product. The author's unsubstantiated opinion is that the total recoil mechanism will be lighter in weight and also less costly to manufacture than a conventional recoil mechanism.

RECOMMENDATIONS

The CFSR system appears to be a feasible and worthwhile concept to pursue further. It is recommended that a working model be built and tested based on the data given in this technical report. Some of the assumed parameters can then be verified or corrected as, for instance, the flow coefficient, the effective bulk modulus, stresses and strains in the metal parts, energy storing capability, and the actual seal friction. The mathematical model can then be corrected accordingly for reuse.

Dr. R. Yalamanchili has suggested making a CFSR mechanism with a cylindrical piston going right through the recoil cylinder. Although it is realized there would be some design problems, the author believes this is a very sound idea. This design would not have the undesirable features of the current concept.

The piston wall would be a little thicker where it enters the cylinder than where it exits, with a step in the thickness in between. The apparent advantage is in having a thick and stiff piston with a very small differential cross sectional area between the ends.

This design could have a long, soft stroke and low initial pressure. Less fluid volume (and weight) would be possible, cavitation could easily be eliminated, and there would be a much lower piston force and latch velocity.

Figure 6 shows this system as an alternate concept with approximate proportions and dimensions.

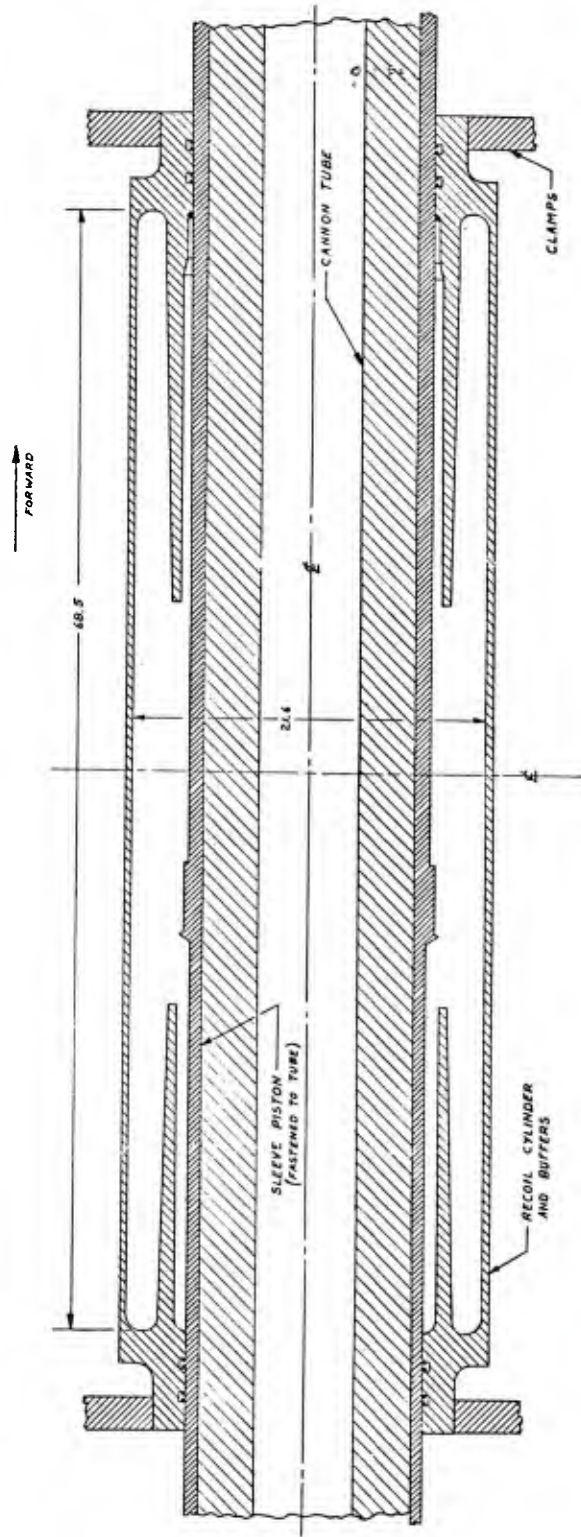


Figure 6. Preliminary CFSR concept study no. 2.
(Dimensions are given in inches)

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APPENDIX A

BREECH FORCE AND BREECH ENERGY

Breech Force

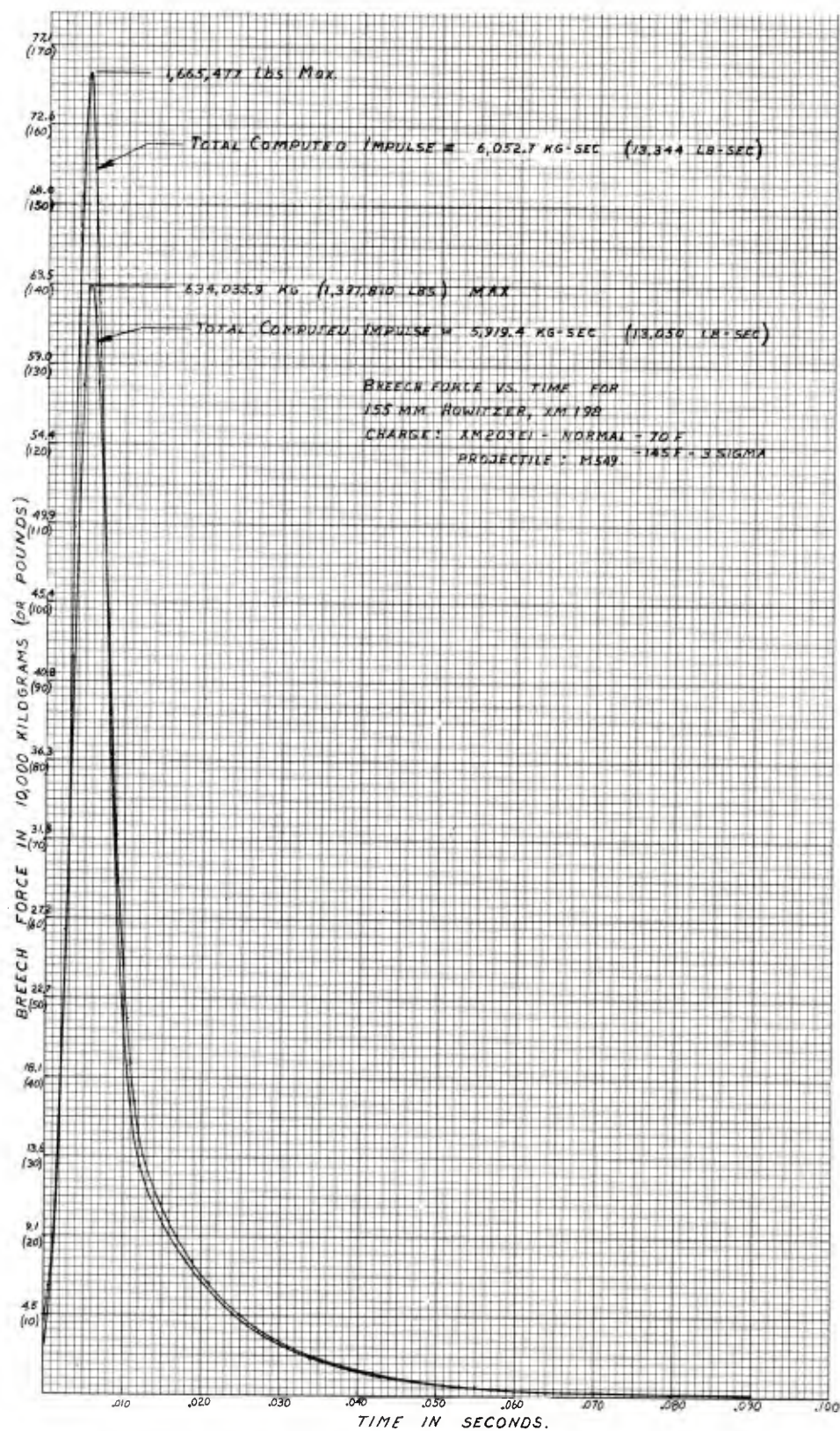
The breech force versus time curve is plotted from the M198 (155 mm) towed howitzer firing data. See page 42.

Breech Energy

The breech energy computations are given below

$$E_{br} = \frac{(I_{mp})^2 g}{2W_R} = \frac{(6052.737)^2 9.81}{2(2948.35)} = 60,948.58 \text{ meter-kg} \quad A1$$

$$E_{br} = \frac{(I_{mp})^2 g}{2W_R} = \frac{(13,344)^2 386.088}{2(6500)} = 5,288,287 \text{ in-lb} \quad A2$$

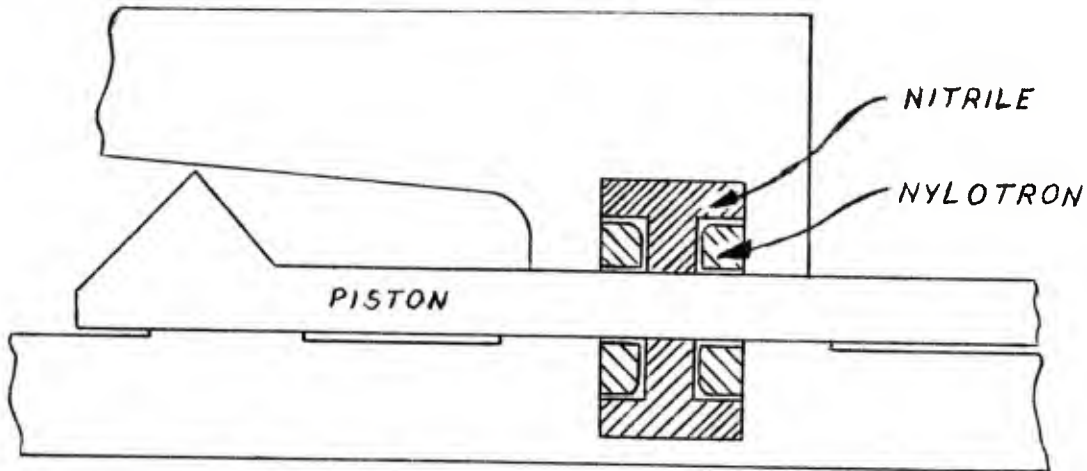


Breech force versus time

APPENDIX B

SEAL FRICTION

Two dynamic T-seals will probably be required, one outside and one inside the piston (see figure). The seals can be supplied by Greene, Tweed & Co., and are made from Nitrile or Buna-N (NBR) rubber with backup rings of nylotron.



Piston seals

An unresolved problem is that the coefficient of friction and the friction force are very uncertain for these materials in this application. The data available at this time does not permit an accurate, detailed analysis of oil pressure and seal friction. The friction formula given below should, however, be adequate for this study.

$$F = \mu_R v_R A_S P \quad (B-1)$$

where:

- F = Axial friction force on the piston
- A_S = Side area of seal backup ring plus edge of T-seal
(Dimensions are taken from the Greene, tweed & Co.'s Palmetto catalog.)

$$\begin{aligned}\mu_R &= 0.50 = \text{coefficient of friction (An approximate figure for} \\ &\quad \text{rubber supplied by RIA Rubber Laboratory.)} \\ \nu_R &= 0.48 = \text{Poisson's ratio for rubber (An approximate figure} \\ &\quad \text{supplied by the RIA Rubber Laboratory.)} \\ P &= \text{Fluid pressure in psi.}\end{aligned}$$

Inside seal:

$$*A_{S_1} = \pi(7.25^2 - 7.00^2) = 11.1919 \text{ in}^2 \quad (\text{B-2})$$

$$*F = 0.50(.48)11.1919P = 2.69 P. \quad (\text{B-3})$$

Outside seal:

$$*A_{S_2} = \pi(8.00^2 - 7.75^2) = 12.37 \text{ in}^2 \quad (\text{B-4})$$

$$*F = 0.50(.48)12.37P = 2.97 P. \quad (\text{B-5})$$

This relationship, variable with pressure, was used in the mathematical model. It is the best approximation available for the seal friction at the present time.

Since the piston will be machined to slide over the inside sleeve and probably will have a fairly close sliding fit in certain areas, it is believed that the inside seal never "sees" the full peak pressure which only lasts a fraction of a second. It was therefore arbitrarily decided to use only 50% of the force on this seal, which accounts for the 0.5 factor in equation 79.

Greene, Tweed and Co., North Wales, PA can supply G-T Ring seals for pressures from zero and up to 6.895×10^7 Pa (10,000 psi) and higher. The computed CFSR pressures are all well within this range.

The final dimensions of the piston and seals are not the exact dimensions used in the friction calculations shown above and used in the mathematical model. There should not, however, be any significant change in the forces.

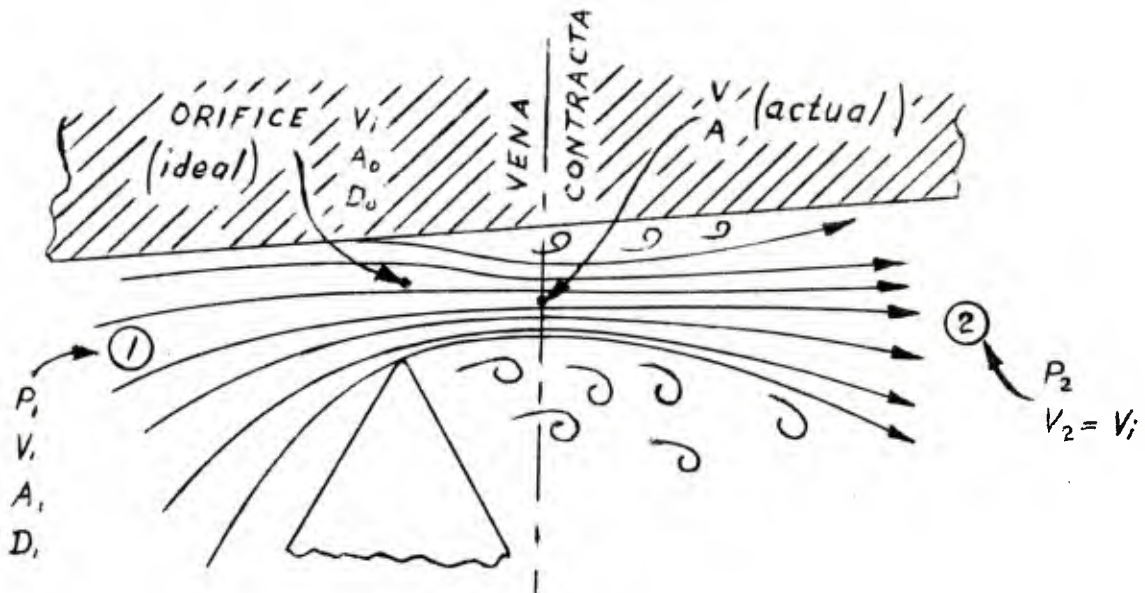
*These formulas are given in English units.

APPENDIX C

FLOW COEFFICIENT

The following information can be found in any textbook on fluid mechanics, but is included here to emphasize the number of physical conditions which the flow coefficient is dependent upon.

Looking at the flow going through an orifice (see figure), some coefficients and relationships are established.



Flow through orifice

The relationship between actual velocity (V) and ideal frictionless velocity (V_i) at "vena contracta" is determined by the coefficient of velocity (C_v), thus:

$$V = C_v V_i \quad (C-1)$$

Vena contracta is the minimum cross section of the jet where it is contracted right outside the orifice on the down-stream side. This contraction is dependent on friction and shape of the orifice and the viscosity of the fluid.

The ratio of the cross-sectional area of a jet (A) at vena contracta to the area at the orifice (A_o) is called the coefficient of contraction (C_c). This relationship can be expressed as follows:

$$A = C_c A_o . \quad (C-2)$$

The ratio of the actual rate of discharge (Q) to the ideal rate of discharge (Q_i), if there were no friction and no contraction, may be defined as the coefficient of discharge (C_d).

Thus:

$$Q = C_d Q_i \quad (C-3)$$

By observing that $Q = AV$ and $Q_i = A_o V_i$, it is seen that:

$$C_d = C_c C_v . \quad (C-4)$$

Writing the Bernoulli's equation thru the orifice in the buffer, it

$$\frac{P_1}{\omega} + \frac{V_1^2}{2g} = \frac{P_2}{\omega} + \frac{V_2^2}{2g} \quad (C-5)$$

(See figure.)

Rearranging this equation to get the flow velocity outside the orifice ($V_2 = V_i$), it becomes

$$V_2 = \sqrt{\frac{2g}{\omega} (P_1 - P_2) + V_1^2} \quad (C-6)$$

and the actual jet velocity is

$$V = C_v \sqrt{\frac{2g}{\omega} (P_1 - P_2) + V_1^2} . \quad (C-7)$$

By applying the continuity equation

$$V_1 A_1 = AV \quad (C-8)$$

$$V_1 = AV/A \quad (C-9)$$

$$V_1 = C_c A_0 V/A \quad (C-10)$$

$$V_1 = C_c V (D_0/D_1)^2 \quad (C-11)$$

and substituting this V_1 value into the Bernoulli equation, it becomes

$$V = \frac{C_v}{\sqrt{1 - C_d^2 (D_0/D_1)^4}} \sqrt{\frac{2g}{\omega} (P_1 - P_2)} \quad (C-12)$$

Since

$$Q = C_c A_0 V, \quad (C-13)$$

$$Q = \frac{C_d A_0}{\sqrt{1 - C_d^2 (D_0/D_1)^4}} \sqrt{\frac{2g}{\omega} (P_1 - P_2)} \quad (C-14)$$

Now to simplify this equation,

$$\frac{C_d}{\sqrt{1 - C_d^2 (D_0/D_1)^4}} = K \quad (C-15)$$

K is called the flow coefficient, and the final equation becomes

$$Q = K A_0 \sqrt{\frac{2g}{\omega} (P_1 - P_2)} \quad (C-16)$$

This is the equation which is used in the mathematical model.

Since there is a variable orifice-- D_o , D_1 , and A_o vary-- K is not a constant. In addition, C_d , C_v , and C_c , which are dependent on orifice shape and friction or drag and viscosity, vary. Viscosity of the fluid is dependent on pressure and temperature. It must also be remembered that the specific weight of the fluid ω varies with pressure and temperature.

Unfortunately, there are no reliable flow coefficients available for this fluid. It was assumed, however, that there is an average constant coefficient for this application which would apply for the given conditions and boundaries, accounting for all the variables including ω . The very best estimate for such a value is,

$$K = 0.95$$

(C-17)

APPENDIX D

BULK MODULUS

Dow Corning 200 (10 cs), a silicon fluid, is used as the working fluid in the recoil mechanism. This liquid has good compressibility and stability and a relatively flat viscosity curve.

However, many of its physical properties are not very well known. There are no precise data available either for the absolute bulk modulus, or for the effective bulk modulus which includes the container--in this case the recoil cylinder. The bulk modulus (β), an essential parameter for the solution of this recoil problem, varies with pressure and temperature over a wide range.

The bulk modulus is defined by the following differential equation:

$$\beta = - V_i \frac{dP}{dv} , \quad (D-1)$$

where

P = Fluid pressure

V_i = Initial fluid volume

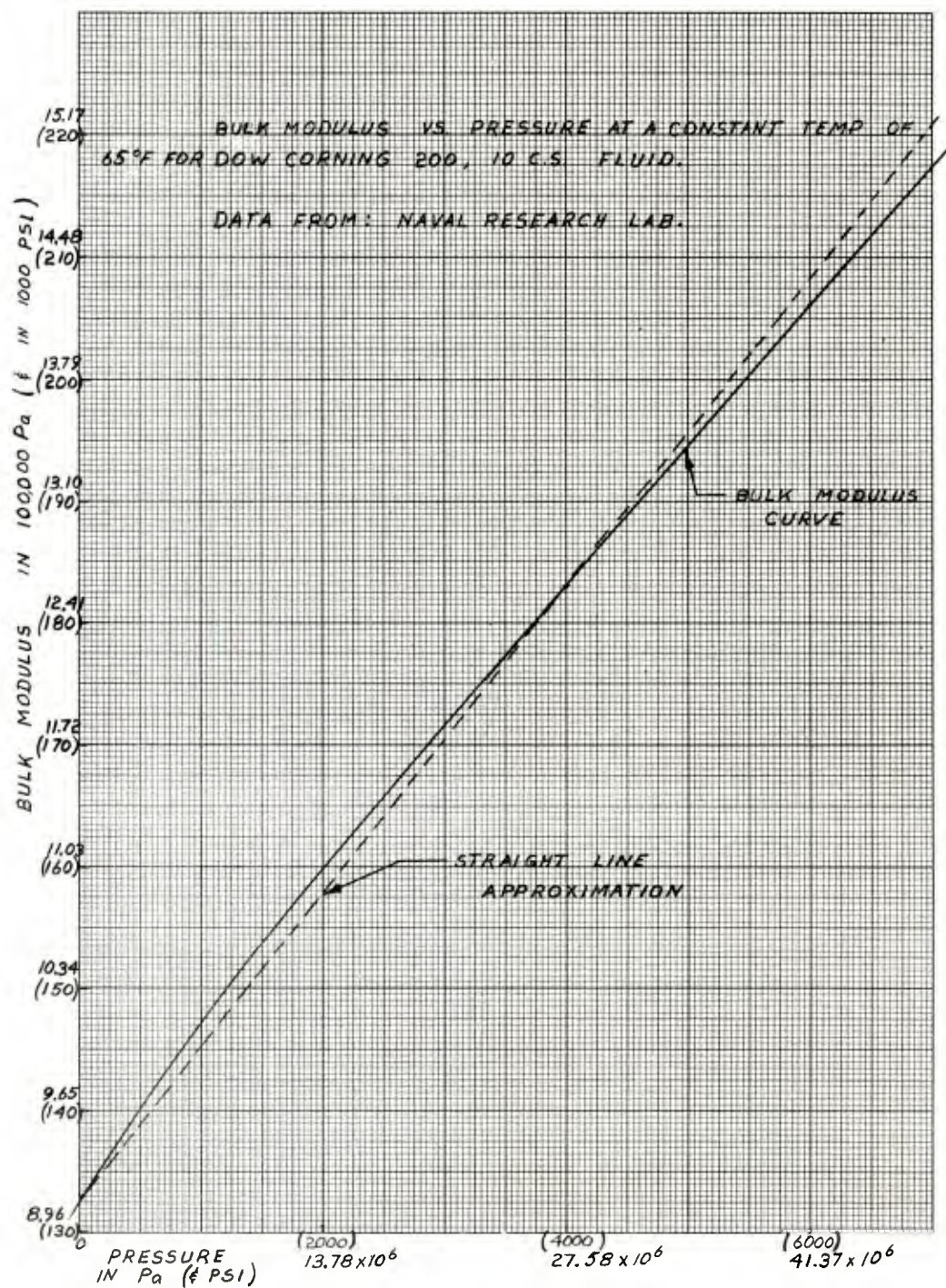
V = Fluid volume at time (t).

The compressibility (k) of the fluid is defined as

$$k = \frac{1}{\beta} , \quad (D-2)$$

A few values of bulk modulus for four different pressures, covering approximately half of the pressure range of the recoil mechanism, were published by the Naval Research Laboratory. Since pressures in a recoil mechanism vary greatly during a cycle, and temperatures vary only a small amount, a curve was plotted for room temperature conditions only (65°F), using the four points available. This curve proved to be practically a straight sloping line, so a straight line was used for the needed β versus P relationship (see figure). The tangent to this curve is 12.67. Thus, the following relationship was used in the mathematical model:

$$\begin{aligned} \beta &= 12.67 P + 9.136 \times 10^8 \\ (\beta &= 12.67 P + 132,500) \end{aligned} \quad (D-3)$$



Bulk modulus versus pressure

APPENDIX E
COMPUTER PRINTOUTS
(TOTAL PROGRAM)

DARCOM MIDWEST SECC	RM9.PRI	A..START	JOB	7994...	2.03.21	PM	15	JUN	77.2T03.K173.KW3PDB70.BENZKOFER	*DARCOM MIDWEST SECC*
DARCOM MIDWEST SECC	RM9.PRI	A..START	JOB	7994...	2.03.21	PM	15	JUN	77.2T03.K173.KW3PDB70.BENZKOFER	*DARCOM MIDWEST SECC*
DARCOM MIDWEST SECC	RM9.PRI	A..START	JOB	7994...	2.03.21	PM	15	JUN	77.2T03.K173.KW3PDB70.BENZKOFER	*DARCOM MIDWEST SECC*

H A S P J O B L O G

\$13.51.19 JOB 7994 - KW3PD870 - RDR ON 13.22.18 - BEGINNING EXEC - INIT 4 - CLASS A
 13.51.21 JOB 7994 IEF4031 KW3PD870 STARTED TIME=13.51.21
 \$13.52.20 JOB 7994 ESTIMATED TIME EXCEEDED
 13.57.46 JOB 7994 IEF4041 KW3PD870 ENDED TIME=13.57.46
 N13.57.47 JOB 7994 END EXECUTION.

----- HASP-II JOB STATISTICS -----

426 CARDS READ

1,214 SYSOUT PRINT RECORDS

0 SYSOUT PUNCH RECORDS

6.49 MINUTES EXECUTION TIME

```

//KW3PDB70 JOB (2703,K173,1,3,9999,.,.,65),BENZKOFER,
// EXEC FORTGCLG,REGION=150K
XXDEFAULTS PROC SYSOUT=A
XAFORT EXEC PGM=IEYF0RT,REGION=106K,DPRTY=(3,3)
XXSYSPRINT DD SYSOUT=&SYSOUT
IEF6531 SUBSTITUTION JCL - SYSOUT=A
XXSYSLIN DD DSN=&&OBJECT,UNIT=ITEL,SPACE=(CYL,(1,1)),
XX DCB=BLKSIZE=800,DISP=(,PASS)
//FORT.SYSIN DD *
IEF2361 ALLOC. FOR KW3PDB70 FORT
IEF2371 0C2 ALLOCATED TO SYSPRINT
IEF2371 114 ALLOCATED TO SYSLIN
IEF2371 090 ALLOCATED TO SYSIN
IEF1421 - STEP WAS EXECUTED - COND CODE 0000
IEF2851 SYS77165.T131819.RV000.KW3PDB70.OBJECT PASSED
IEF2851 VOL SER NOS= IFSEOS.
IEF3731 STEP /FORT / START 77166.1351
IEF3741 STEP /FORT / STOP 77166.1351 CPU 0MIN 16.82SEC MAIN 100K CC= 0

**STEP FORT **JOB KW3PDB70*****
*RESOURCE- CORE(K) DISK(10) TAPE(10)--UNITS(U) IN-HASP(10)-OUT OTHER(10) CPU TIME(C) STEP TIME(T) *
*USAGE- 150 53 0 0 404 553 0 00:00:16.82 00:00:25.87 *
*****

XXLKED EXEC PGM=IEWLF80,PARM=LIST,LET,XREF,REGION=100K,
XX COND=(4,LT,FORT),DPRTY=(3,3)
XXSYSPRINT DD SYSOUT=&SYSOUT
IEF6531 SUBSTITUTION JCL - SYSOUT=A
XXSYSLIN DD DSN=NONO.FORTLIB,DISP=SHR
XXSYSLIN DD DSN=&&OBJECT,DISP=(OLD,DELETE)
XX DD DDNAME=SYSIN
XXSVSUT1 DD UNIT=ITEL,SPACE=(CYL,(1,1))
XXSVSLMOD DD DSN=&&LOAD(MAIN),UNIT=ITEL,SPACE=(CYL,(1,3,1)),
XX DISP=(,PASS)
IEF2361 ALLOC. FOR KW3PDB70 LKED
IEF2371 0C2 ALLOCATED TO SYSPRINT
IEF2371 115 ALLOCATED TO SYSLIB
IEF2371 114 ALLOCATED TO SYSLIN
IEF2371 114 ALLOCATED TO SVSUT1
IEF2371 114 ALLOCATED TO SVSLMOD
IEF1421 - STEP WAS EXECUTED - COND CODE 0000
IEF2851 SYS1.FORTLIB KEPT
IEF2851 VOL SER NOS= IFSEPP.
IEF2851 SYS77165.T131819.RV000.KW3PDB70.OBJECT DELETED
IEF2851 VOL SER NOS= IFSEOS.
IEF2851 SYS77165.T131819.RV000.KW3PDB70.R0004992 DELETED
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IEF2851 SYS77165.T131819.RV000.KW3PDB70.LOAD PASSED
IEF2851 VOL SER NOS= IFSEOS.
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IEF3741 STEP /LKED / STOP 77166.1352 CPU 0MIN 02.24SEC MAIN 98K CC= 0

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*USAGE- 150 337 0 0 163 0 00:00:02.24 00:00:15.57 *
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```



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XXGO      EXEC PGM=*.LKED.SYSLMOD,COND=((4,LT,FORT),(4,LT,LKED)),
XX          DPTY=(3,3)
XXFT05F001 DD      DDNAME=SYSIN
XXFT06F001 DD      SYSOUT=ASYSOUT
IEF6531 SUBSTITUTION JCL - SYSOUT=A
XXFT07F001 DD      SYSOUT=B
//GO.SYSIN DD
//
IEF2361 ALLOC. FOR KW3PDB70 GO
IEF2371 114 ALLOCATED TO PGM=*.DD
IEF2371 090 ALLOCATED TO FT05F001
IEF2371 0C2 ALLOCATED TO FT06F001
IEF2371 1A2 ALLOCATED TO FT07F001
IEF1421 - STEP WAS EXECUTED - COND CODE 0000
IEF2851 SYS77165.T131819.RV000.KW3PDB70.LOAD
IEF2851 VOL SER NOS= IFSEOS.
IEF3731 STEP /GO / START 77166.1352
IEF3741 STEP /GO / STOP 77166.1357 CPU 3MIN 19.59SEC MAIN 48K CC= 0
PASSED

*STEP GO
*J08 KW3PDB70*****
*RESOURCE= CORE(K) DISK(IO) TAPE(IO)--UNITS(U) IN-HASP(IO)-OUT OTHER(IO) CPU TIME(C) STEP TIME(T) *
*USAGE= 150 0 0 20 409 0 00:03:19.59 00:05:34.17 *
*****

IEF2851 SYS77165.T131819.RV000.KW3PDB70.LOAD
IEF2851 VOL SER NOS= IFSEOS.
IEF3751 J08 /KW3PDB70/ START 77166.1351
IEF3761 J08 /KW3PDB70/ STOP 77166.1357 CPU 3MIN 38.65SEC

*J08 KW3PDB70**2T03**X173**RENZKOFER*****
*UNITS= CORE(K*T) DISK(IO) TAPE(IO) TAPE(U*T) HASPI(IO) HASPO(IO) OTHER(IO) CPU(C) TOTAL *
* COSTS= $1.97 $0.12 $0.00 $0.00 $0.00 $0.00 $0.00 $13.36 $15.45 *
*****

```

```

0001 REAL L1,L2,L3,L4,MR,N,NU,I,J
0002 EXTERNAL F7,F8,F9
0003 DIMENSION TIME(81),BFORCE(81)
0004 COMMON/BLK1/H,HH
0005 COMMON/BLK2/DI,F,HI,I,J,R,S,TI,U
0006 COMMON/BLK3/IP,P81,P82,P83
0007 COMMON/BLK4/A,H,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0008 COMMON/BLK5/A1,A2,A3,A4,A5,R1,R2,R3,R4,R5
0009 NAMELIST/NAM/R1,R2,R3,R4,R5,A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,
1E,N,NU,V0,WR,GAMMA
0010 NAMELIST/NAM1/TIME,BFORCE
0011 READ(5,NAM)
0012 WRITE(6,NAM)
0013 READ(5,NAM1)
0014 WRITE(6,NAM1)
0015 MR=WR/G
0016 DI=0.
0017 U=0.
0018 F=0.
0019 R=1.
0020 HI=0.
0021 J=0.
0022 TI=-1.
0023 S=-1.
0024 I=0.
0025 IMISFR=0
0026 INORM=1
0027 IRBUFF=0
0028 VE=240.
0029 ICHECK=0
0030 IPRESS=0
0031 T=0.
0032 P81=0.
0033 P82=0.
0034 P83=0.
0035 P2=3000.
0036 P1=P2
0037 P3=3000.
L4=L1+L2+L3
L4=R5.
DELA=3.1416*L4*P2*A/(3.1416*L4*E/((8**2+(1.-NU)*A**2)/(8**2-
1A**2)+NU))
DELD=-3.1416*L4*P2*D/(3.1416*L4*E/((0**2+(1.-NU)*C**2)/(0**2-
1C**2)-NU))
DELA0=0.
DELD0=0.
A1=3.1416*(R2**2-R3**2)
A2=3.1416*(R1**2-R3**2)
A3=3.1416*(R3**2-R5**2)
A4=3.1416*(R3**2-R4**2)
A5=3.1416*(R4**2-R5**2)
X=L3
XD=0.

```

C

```

0050 H=.00005
0051 HH=H/2.
0052 SUM=L2+L3
0053 IP=1
0054 KOUNT=0
0055 EF=0.
0056 EC=0.
0057 IF(INORM.EQ.1) GO TO 3
0058 IF(INORM.EQ.0.AND.IMISFR.EQ.1) GO TO 12
0059 1 CALL LINEAR(T,TIME,BFORCE,PB1)
0060 T=T+HH
0061 CALL LINEAR(T,TIME,BFORCE,PB2)
0062 T=T+HH
0063 CALL LINEAR(T,TIME,BFORCE,PB3)
0064 IP=1
0065 3 CALL KUTTA(X,XD,P3,P2)
0066 XDD=F1(X,XD,P3,P2)
0067 CALL KUTTA2(DELA,DELD,DELAD,DELDD,P2)
0068 DELADD=F5(DELA,DELAD,P2)
0069 DELDD=F6(DELD,DELDD,P2)
0070 CALL KUTTA1(X,XD,P2,P2,P3,DELA,DELAD,DELD,DELDD)
0071 P10=F2(X,XD,P1,P2)
0072 P20=F3(X,XD,P1,P2,P3,DELA,DELAD,DELD,DELDD)
0073 P30=F4(X,XD,P2,P3)
0074 IF(X.LT.SUM) GO TO 13
0075 DI=1.
0076 U=1.
0077 F=1.
0078 GO TO 13
0079 12 T=T+H
0080 CALL KUTTA(X,XD,P3,P1)
0081 XDD=F1(X,XD,P3,P1)
0082 IF(X.LT.SUM) GO TO 16
0083 DI=1.
0084 U=1.
0085 F=1.
0086 16 CALL KUTTA2(DELA,DELD,DELAD,DELDD,P2)
0087 DELADD=F5(DELA,DELAD,P2)
0088 DELDD=F6(DELD,DELDD,P2)
0089 CALL KUTTA1(X,XD,P1,P2,P3,DELA,DELAD,DELD,DELDD)
0090 P10=F2(X,XD,P1,P2)
0091 P20=F3(X,XD,P1,P2,P3,DELA,DELAD,DELD,DELDD)
0092 P30=F4(X,XD,P2,P3)
0093 IF(X.GT.SUM) P3=P2
0094 IF(X.GT.SUM) GO TO 14
0095 P1=P2
0096 P3=P2
0097 DI=0.
0098 U=0.
0099 F=0.
0100 GO TO 14
0101 13 IF(IRRUFF.EQ.1) GO TO 11
0102 IF(X.GT.SUM) GO TO 19

```



```

0103      DI=0.
0104      U=0.
0105      F=0.
0106      P1=P2
0107      P3=P2
0108
0109      GO TO 14
0110
0111      19 P3=P2
0112      14 SIGT=(R**2+A**2)/(B**2-A**2)*P2
0113      SIGAX=A**2*P2/(B**2-A**2)
0114      SIGR=-P2
0115      SIGE=.5*SQRT((SIGT-SIGR)**2+(SIGR-SIGAX)**2+(SIGAX-SIGT)**2)
0116      V1=3.1416*(A**2-D**2)*L4+V0
0117      KOUNT=KOUNT+1
0118      IF(P2.LE.0.) GO TO 20
0119      V=3.1416*(A*DELA)**2-(D*DELD)**2)*L4+A5*(L4-X)
0120      EF=EF-F9(V,P2,P20,N)*H
0121      EC=EC+P.*3.1416*L4*(F7(A*DELA*DELD,P2)+F8(D*DELD*DELD,P2))*H
0122      20 IF(KOUNT.LT.40) GO TO 7
0123      KOUNT=0
0124
0125      PRINT 5,I,X,XD,XDD,P1,P1D,P2,P2D,P3,P3D
0126      5 FORMAT(2X,'T=',F6.4,'X=',F6.3,'XDD=',F9.2,'X',XDD=',F9.1,'X,
1'P1=',F9.2,'X,P1D=',F10.1,'X,P2=',F9.2,'X,P2D=',F10.1,'X,
2'P3=',F9.2,'X,P3D=',F9.1)
0127      PRINT 2,DELA,DELD,DELD,DELD,SIGT,SIGAX,SIGR,SIGE
0128      2 FORMAT(2X,'DELA=',F7.4,'X,'DELD=',F7.4,'X,'DELD=',F7.4,'X,
1'DELDD=',F7.4,'X,'SIGT=',F10.1,'X,'SIGAX=',F10.1,'X,
2'SIGR=',F10.1,'X,'SIGE=',F10.1)
0129      PRINT 2,DELA,DELD,DELD,DELD,SIGT,SIGAX,SIGR,SIGE,EC,EF
0130      2 FORMAT(2X,'DELA=',F7.4,'X,'DELD=',F7.4,'X,
1'SIGT=',F10.1,'X,'SIGAX=',F10.1,'X,
2'SIGR=',F10.1,'X,'SIGE=',F10.1,'X,
3'EC=',F10.1,'X,'EF=',F15.1)
0131      7 IF(INORM.EQ.0.AND.IMISFR.EQ.0) GO TO 10
0132      IF(ICHECK.GE.1) GO TO 4
0133      T=T+H
0134      GO TO 3
0135      4 ICHECK=ICHECK+1
0136      IF(ICHECK.EQ.1) TT=0.
0137      CALL LINEAR(TT,TIME,BFORCE,PB1)
0138      TT=TT+HH
0139      CALL LINEAR(TT,TIME,BFORCE,PB2)
0140      TT=TT+HH
0141      CALL LINEAR(TT,TIME,BFORCE,PB3)
0142      IP=1
0143      T=T+H
0144      HI=1.
0145      IF(XD.GE.0.) GO TO 8

```

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MAIN

FORTRAN IV G LEVEL 21

```

0144 S=1.
0145 GO TO 9
0146 8 S=-1.
0147 TI=-1.
0148 9 IF(X.GT.L3.AND.IRBUFF.EQ.0) GO TO 3
0149 I=-1.
0150 J=1.
0151 R=0.
0152 TI=1.
0153 IRBUFF=1
0154 IF(XD.GT.0..AND.X.GT.L3) GO TO 6
0155 GO TO 3
0156 10 IF(XD.LI.0..AND.IPRESS.EQ.1.AND.X.LT.L3) GO TO 1
0157 IF(XD.LI.0..AND.IPRESS.EQ.1.AND.X.GE.L3) GO TO 18
0158 IF(XD.LI.0..AND.IPRESS.EQ.0.AND.X.GT.L3) GO TO 17
0159 IF(XD.GT.0..AND.IPRESS.EQ.0.AND.X.GE.L3) GO TO 18
0160 IF(XD.GT.0..AND.IPRESS.EQ.1.AND.X.LT.L3) GO TO 1
0161 IF(XD.GT.0..AND.IPRESS.EQ.1.AND.X.GE.L3) GO TO 6
0162 17 IPRESS=1
0163 GO TO 1
0164 18 X=L3
0165 XD=0.
0166 GO TO 1
0167 15 IF(XD.GT.0.) GO TO 12
0168 6 PRINT 21,EC,EF
0169 21 FORMAT(5X,'EC=',F30.5,5X,'EF=',F30.5)
0170 CALL EXIT
0171 END

```

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MAIN

FORTRAN IV G LEVEL 21

```

*OPTIONS IN EFFECT* NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = MAIN , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 171,PROGRAM SIZE = 5500
*STATISTICS* NO DIAGNOSTICS GENERATED

```

```

0001 FUNCTION AF(X)
0002 REAL L1,L2,L3,L4,MR,N,NU,I,J
0003 COMMON/BLK4/A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0004 COMMON/BLK5/A1,A2,A3,A4,A5,R1,R2,R3,R4,R5
0005 GF=R2-R3
0006 AF=A1+3.1416*GF**2*(L2+L3)**2/L1**2+2.*3.1416*GF*R2*(L2+L3)/L1-
1(2.*3.1416*GF**2*(L2+L3)/L1**2+2.*3.1416*GF*R2/L1)*X+
23.1416*GF**2*X**2/L1**2
0007 RETURN
0008 END

```

```

*OPTIONS IN EFFECT* NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = AF , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 8,PROGRAM SIZE = 484
*STATISTICS* NO DIAGNOSTICS GENERATED

```

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AR

FORTRAN IV G LEVEL 21

```
0001 FUNCTION AR(X)
0002 REAL L1,L2,L3,L4,MR,N,NU,I,J
0003 COMMON/BLK4/A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0004 COMMON/BLK5/A1,A2,A3,A4,A5,R1,R2,R3,R4,R5
0005 GR=R1-R3
0006 AR=2.*3.1416*GR*R3*X/L3+3.1416*GR**2*X**2/L3**2
0007 RETURN
0008 END
```

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AR

FORTRAN IV G LEVEL 21

```
*OPTIONS IN EFFECT* NOID,ERCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = AR , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 6,PROGRAM SIZE = 380
*STATISTICS* NO DIAGNOSTICS GENERATED
```

0001
 0002
 0003
 0004
 0005
 0006

FORTRAN IV G LEVEL 21 QF DATE = 77166 13/51/23 PAGE 0001

```

    FUNCTION QF(P1,P2,X)
    REAL L1,L2,L3,L4,MR,N,NU,I,J
    COMMON/BLK4/A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
    QF=AF(X)*CD*SQRT(2.*G*ABS(P1-P2)/RHOF)*SGN(P1-P2)
    RETURN
    END
  
```

0001
 0002
 0003
 0004
 0005
 0006

FORTRAN IV G LEVEL 21 QF DATE = 77166 13/51/23 PAGE 0002

```

    *OPTIONS IN EFFECT* NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
    *OPTIONS IN EFFECT* NAME = QF
    *STATISTICS* SOURCE STATEMENTS = 55
    *STATISTICS* SOURCE STATEMENTS = 6,PROGRAM SIZE = 478
    *STATISTICS* NO DIAGNOSTICS GENERATED
  
```

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QR

FORTRAN IV G LEVEL 21

```
0001 FUNCTION QR(P2,P3,X)
0002 REAL L1,L2,L3,L4,MR,N,NU,I,J
0003 COMMON/BLK4/A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0004 QR=AR(X)*CD*SQRT(2.*G*ABS(P3-P2)/RHOF)*SGN(P3-P2)
0005 RETURN
0006 END
```

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QR

FORTRAN IV G LEVEL 21

```
*OPTIONS IN EFFECT* NOID,ERCDCIC,SOURCE,NOLIST,NODECK,LOAD,NOMAM
*OPTIONS IN EFFECT* NAME = QR , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 6,PROGRAM SIZE = 478
*STATISTICS* NO DIAGNOSTICS GENERATED
```

FORTRAN IV G LEVEL 21 F1 DATE = 77166 13/51/23 PAGE 0001

```

0001 FUNCTION F1(X,XD,P3,P1)
0002 REAL L1,L2,L3,L4,MR,N,NU,I,J
0003 COMMON/BLK2/DI,F,HI,I,J,R,S,TI,U
0004 COMMON/BLK4/A,B,C,D,G,CD,PHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0005 COMMON/BLK5/A1,A2,A3,A4,A5,R1,R2,R3,R4,R5
0006 IF(XD.GT.0.) S=-1.
0007 IF(XD.LE.0.) S=1.
0008 F1=1./MR*(S*(2.69*.5*P3+2.97*P1)+P3*A3-P1*A4-HI*PBF(XXX))-
0009 14R*G*SIN(GAMMA))
0010 RETURN
      END

```

FORTRAN IV G LEVEL 21 F1 DATE = 77166 13/51/23 PAGE 0002

```

*OPTIONS IN EFFECT* NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = F1 , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 10,PROGRAM SIZE = 618
*STATISTICS* NO DIAGNOSTICS GENERATED

```

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23

FORTRAN IV G LEVEL 21

```

0001 FUNCTION F2(X,XD,P1,P2)
0002 REAL L1,L2,L3,L4,MR,N,NU,I,J
0003 COMMON/BLK2/DI,F,HI,I,J,R,S,TI,U
0004 COMMON/BLK4/A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0005 COMMON/BLK5/A1,A2,A3,A4,A5,A6,R1,R2,R3,R4,R5
0006 F2=U*(N*P1+132500.)*(A4*XD-QF(P1,P2,X))/(A4*AF(X))*(L1+L2+L3-X)
0007 RETURN
0008 END

```

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F2

FORTRAN IV G LEVEL 21

```
*OPTIONS IN EFFECT* NOID,EBCDIC,SOURCE,NOLIST,NOCHECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = F2 , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 8,PROGRAM SIZE = 520
*STATISTICS* NO DIAGNOSTICS GENERATED
```



```

0001 FUNCTION F3(X,XD,P1,P2,P3,DELA,DELAD,DELD,DELDD)
0002 REAL L1,L2,L3,L4,MR,N,NU,I,J
0003 COMMON/RLK2/DI,F,HI,I,J,R,S,TI,U
0004 COMMON/RLK4/A,R,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0005 COMMON/RLK5/A1,A2,A3,A4,A5,R1,R2,R3,R4,R5
0006 BULK=(N*P2+132500.)
0007 IF(X.CF.L3) REL.
0008 F3=HULK*(1.*(2.*3.1416*L4*(A+DELA)*DELAD+(D+DELD)*DELDD)+R*A5*
0009 1XD)+U*(A4*XD-QF(P1,P2,X))-J*(A4*XD+QR(P2,P3,X))/(3.1416*L4*
0010 2((A+DELA)**2-(D+DELD)**2)+V0*A5*(X-(L1+L2+L3))-J*(A3+AR(X))*X+
0011 3U*(A4+AF(X))*(X-(L1+L2+L3))
    RETURN
    END

```

```

*OPTIONS IN EFFECT* NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = F3 , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 11,PROGRAM SIZE = 970
*STATISTICS* NO DIAGNOSTICS GENERATED

```

FORTRAN IV G LEVEL 21 F4 DATE = 77166 13/51/23 PAGE 0001

```

0001      FUNCTION F4(X,XD,P2,P3)
0002      REAL L1,L2,L3,L4,MR,N,NU,I,J
0003      COMMON/RLK2/DI,F,HI,I,J,R,S,TI,U
0004      COMMON/RLK4/A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0005      COMMON/RLK5/A1,A2,A3,A4,A5,R1,R2,R3,R4,R5
0006      BULK=N*P3+132500.
0007      F4=-J*BULK*(A3*XD*QR(P2,P3,X))/((A3*AR(X))*X)
0008      RETURN
0009      END

```

FORTRAN IV G LEVEL 21 F4 DATE = 77166 13/51/23 PAGE 0002

```

*OPTIONS IN EFFECT*   NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT*   NAME = F4, LINECNT = 55
*STATISTICS*          SOURCE STATEMENTS = 9,PROGRAM SIZE = 514
*STATISTICS*          NO DIAGNOSTICS GENERATED

```

0001
 0002
 0003
 0004
 0005
 0006

FUNCTION F5(DELA,DELAD,P2)
 REAL L1,L2,L3,L4,MR,N,NU,I,J
 COMMON/RLK4/A,B,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
 F5=1/(RHOF*(RHOF-2*(A**2+B**2+C**2)+15*(D**2+E**2)+NU))
 RETURN
 END

FORTTRAN IV G LEVEL 21 F5 DATE = 77166 13/51/23 PAGE 0001

0001
 0002
 0003
 0004
 0005
 0006

OPTIONS IN EFFECT NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
 OPTIONS IN EFFECT NAME = F5
 STATISTICS SOURCE STATEMENTS = 55
 STATISTICS NO DIAGNOSTICS GENERATED 6*PROGRAM SIZE = 472

FORTTRAN IV G LEVEL 21 F5 DATE = 77166 13/51/23 PAGE 0002

FORTTRAN IV G LEVEL 21 F6 DATE = 77166 13/51/23 PAGE 0001

```
0001 FUNCTION F6(DELD,DELD,P2)
0002 REAL L1,L2,L3,L4,MR,N,NU,I,J
0003 COMMON/BLK4/A,R,C,D,G,CD,RHOF,RHOS,L1,L2,L3,L4,E,N,NU,V0,MR,GAMMA
0004 F6=1./ (RHOS*(D**2-C**2))*(-2.*D*p2-2.*E*DELD/((D**2*(1.-NU)+
      1C**2)/(D**2-C**2)-NU))
0005 RETURN
0006 END
```

FORTTRAN IV G LEVEL 21 F6 DATE = 77166 13/51/23 PAGE 0002

```
*OPTIONS IN EFFECT* NOID,ERC,DIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = F6 , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 6,PROGRAM SIZE = 472
*STATISTICS* NO DIAGNOSTICS GENERATED
```

FORTRAN IV G LEVEL 21
 0001
 0002
 0003
 0004
 FUNCTION F7(A,DELA,DELAD,P2)
 F7=(A*DELA)*DELAD*P2
 RETURN
 END
 DATE = 77166
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FORTRAN IV G LEVEL 21
 F7
 OPTIONS IN EFFECT NOID,ERCOIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
 OPTIONS IN EFFECT NAME = F7
 STATISTICS SOURCE STATEMENTS = 55
 STATISTICS NO DIAGNOSTICS GENERATED
 4,PROGRAM SIZE = 374
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 PAGE 0002

FORTRAN IV G LEVEL 21 F8 DATE = 77166 13/51/23 PAGE 0001

0001 FUNCTION F8(D,DELD,DELOD,P2)
0002 F8=(D*DELD)*DELOD*P2
0003 RETURN
0004 END

FORTRAN IV G LEVEL 21 F8 DATE = 77166 13/51/23 PAGE 0002

OPTIONS IN EFFECT NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
OPTIONS IN EFFECT NAME = F8 , LINECNT = 55
STATISTICS SOURCE STATEMENTS = 4,PROGRAM SIZE = 374
STATISTICS NO DIAGNOSTICS GENERATED

FORTRAN IV G LEVEL 21

F9

0001
0002
0003
0004
0005

FUNCTION F9(V,P2,P2D,N)
REAL L1,L2,L3,L4,MR,N,N1,I,J
F9=V*P2*P2D/(N*P2*132500.)
RETURN
END

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FORTRAN IV G LEVEL 21

F9

DATE = 77166

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OPTIONS IN EFFECT NO10,ERCDCIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
OPTIONS IN EFFECT NAME = F9 , LINECNT = 55
STATISTICS SOURCE STATEMENTS = 5,PROGRAM SIZE = 388
STATISTICS NO DIAGNOSTICS GENERATED


```

0001 SURROUTINE LINEAR(A,X,Y,VV)
0002 DIMENSION X(81),Y(81)
0003 I=1
      C 1 IF(Y(I+1) .LT. Y(I)) GO TO 100
      C USE FOLLOWING IF AS Y INCREASES X INCREASES
0004 10 IF(A-X(I))3,2,2
      C USE FOLLOWING IF AS Y INCREASES X DECREASES
      C 100 IF(A-X(I))2,2,3
0005 2 I=I+1
0006 GO TO 10
0007 3 I=I-1
0008 VV=Y(I)*(A-X(I+1))/(X(I)-X(I+1))+Y(I+1)*(A-X(I))/(X(I+1)-X(I))
0009 RETURN
0010 END

```

```

*OPTIONS IN EFFECT* NOID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = LINEAR , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 10,PROGRAM SIZE = 506
*STATISTICS* NO DIAGNOSTICS GENERATED

```

FORTRAN IV G LEVEL 21

```

0001 FUNCTION SGN(XX)
0002 IF(XX) 1,2,2
0003 1 SGN=-1.
0004 GO TO 3
0005 2 SGN=1.
0006 3 RETURN
0007 END

```

FORTRAN IV G LEVEL 21

```
*OPTIONS IN EFFECT* NOID,ERCOIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = SON , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 7,PROGRAM SIZE = 346
*STATISTICS* NO DIAGNOSTICS GENERATED
```

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KUTTA

FORTRAN IV G LEVEL 21

```

0001 SURROUTINE KUTTA(X,XD,P1,P2)
0002 COMMON/RLK1/H,HH
0003 AK1=H*F1(X,XD,P1,P2)
0004 ZX=X+HH*XD+H*AK1/8.
0005 ZXD=XD+AK1/2.
0006 AK2=H*F1(ZX,ZXD,P1,P2)
0007 ZXD=XD+AK2/2.
0008 AK3=H*F1(ZX,ZXD,P1,P2)
0009 ZX=X+H*XD+HH*AK3
0010 ZXD=XD+AK3
0011 AK4=H*F1(ZX,ZXD,P1,P2)
0012 X=X+H*(XD+(AK1+AK2+AK3)/6.)
0013 XD=XD+(AK1+2.*(AK2+AK3)+AK4)/6.
0014 RETURN
0015 END
    
```

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DATE = 77166

KUTTA

FORTRAN IV G LEVEL 21

```

*OPTIONS IN EFFECT* NOID,ERCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = KUTTA , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 15,PROGRAM SIZE = 690
*STATISTICS* NO DIAGNOSTICS GENERATED
    
```

```

0001 FUNCTION PBF(XXX)
0002 COMMON/RLK3/IP,PB1,PB2,PB3
0003 IF(IP.GT.1) GO TO 1
0004 PRF=PB1
0005 IP=IP+1
0006 RETURN
0007 1 IF(IP.GT.2) GO TO 2
0008 PBF=PB2
0009 IP=IP+1
0010 RETURN
0011 2 IF(IP.GT.3) GO TO 3
0012 PRF=PB2
0013 IP=IP+1
0014 RETURN
0015 3 PBF=PB3
0016 4 RETURN
0017 END

```

```

*OPTIONS IN EFFECT* NOID,ERCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = PBF , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 17,PROGRAM SIZE = 482
*STATISTICS* NO.DIAGNOSTICS GENERATED

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FORTRAN IV G LEVEL 21 13/51/23 DATE = 77166 PAGE 0001

```
0001 SURROUTINE DQG4(XL,XU,FCT,Y)
0002 A=.5*(XU+XL)
0003 R=XU-XL
0004 C=.43056815579702629*B
0005 Y=.17392742256872693*(FCT(A+C)+FCT(A-C))
0006 C=.16999052179242813*B
0007 Y=R*(Y+.32607257743127307*(FCT(A+C)+FCT(A-C)))
0008 RETURN
0009 END
```

FORTRAN IV G LEVEL 21 13/51/23 DATE = 77166 PAGE 0002

```
*OPTIONS IN EFFECT* NOID,ERCDC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = DQG4 , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 9,PROGRAM SIZE = 586
*STATISTICS* NO DIAGNOSTICS GENERATED
```

```

0001 SUBROUTINE KUTTAL(X,XD,P1,P2,P3,DELA,DELAD,DELD,DELODD)
0002 COMMON/RLK1/H,MH
0003 AK1=H*F2(X,XD,P1,P2)
0004 AL1=H*F3(X,XD,P1,P2,P3,DELA,DELAD,DELD,DELODD)
0005 AM1=H*F4(X,XD,P2,P3)
0006 ZP1=P1+AK1/2.
0007 ZP2=P2+AL1/2.
0008 ZP3=P3+AM1/2.
0009 AK2=H*F2(X,XD,ZP1,ZP2)
0010 AL2=H*F3(X,XD,ZP1,ZP2,ZP3,DELA,DELAD,DELD,DELODD)
0011 AM2=H*F4(X,XD,ZP2,ZP3)
0012 ZP1=P1+AK2/2.
0013 ZP2=P2+AL2/2.
0014 ZP3=P3+AM2/2.
0015 AK3=H*F2(X,XD,ZP1,ZP2)
0016 AL3=H*F3(X,XD,ZP1,ZP2,ZP3,DELA,DELAD,DELD,DELODD)
0017 AM3=H*F4(X,XD,ZP2,ZP3)
0018 ZP1=P1+AK3
0019 ZP2=P2+AL3
0020 ZP3=P3+AM3
0021 AK4=H*F2(X,XD,ZP1,ZP2)
0022 AL4=H*F3(X,XD,ZP1,ZP2,ZP3,DELA,DELAD,DELD,DELODD)
0023 AM4=H*F4(X,XD,ZP2,ZP3)
0024 P1=P1+1./6.*(AK1+2.*AK2+2.*AK3+AK4)
0025 P2=P2+1./6.*(AL1+2.*AL2+2.*AL3+AL4)
0026 P3=P3+1./6.*(AM1+2.*AM2+2.*AM3+AM4)
0027 RETURN
0028 END

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```

*OPTIONS IN EFFECT* NOID,ERCDCIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = KUTTAL , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 28,PROGRAM SIZE = 1308
*STATISTICS* NO DIAGNOSTICS GENERATED

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FORTRAN IV G LEVEL 21 KUTTA2 DATE = 77166 13/51/23 PAGE 0001

```

0001 SURROUTINE KUTTA2(DELA,DELO,DELAD,DELDD,P2)
0002 COMMON/RLK1/H,HH
0003 AK1=H*F5(DELA,DELAD,P2)
0004 AL1=H*F6(DELO,DELDD,P2)
0005 ZDELA=DELA+HH*DELAD+H*AK1/8.
0006 ZDELO=DELO+HH*DELDD+H*AL1/8.
0007 ZDELAD=DELAD+AK1/2.
0008 ZDELDD=DELDD+AL1/2.
0009 AK2=H*F5(7*DELA,ZDELAD,P2)
0010 AL2=H*F6(7*DELO,ZDELDD,P2)
0011 ZDELAD=DELAD+AK2/2.
0012 ZDELDD=DELDD+AL2/2.
0013 AK3=H*F5(ZDELA,ZDELAD,P2)
0014 AL3=H*F6(ZDELO,ZDELDD,P2)
0015 ZDELA=DELA+H*DELAD+HH*AK3
0016 ZDELO=DELO+H*DELDD+HH*AL3
0017 ZDELAD=DELAD+AK3
0018 ZDELDD=DELDD+AL3
0019 AK4=H*F5(ZDELA,ZDELAD,P2)
0020 AL4=H*F6(ZDELO,ZDELDD,P2)
0021 DELA=DELA+H*(DELAD+(AK1+AK2+AK3)/6.)
0022 DELO=DELO+H*(DELDD+(AL1+AL2+AL3)/6.)
0023 DELAD=DELAD*(AK1+2.*(AK2+AK3)+AK4)/6.
0024 DELDD=DELDD*(AL1+2.*(AL2+AL3)+AL4)/6.
0025 RETURN
0026 END

```

FORTRAN IV G LEVEL 21 KUTTA2 DATE = 77166 13/51/23 PAGE 0002

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*OPTIONS IN EFFECT* NOID,ERCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = KUTTA2 , LINECNT = 55
*STATISTICS* SOURCE STATEMENTS = 26,PROGRAM SIZE = 1012
*STATISTICS* NO DIAGNOSTICS GENERATED
*STATISTICS* NO DIAGNOSTICS THIS STEP

```


80

CONTROL SECTION			ENTRY					
NAME	ORIGIN	LENGTH	NAME	LOCATION	NAME	LOCATION	NAME	LOCATION
MAIN	00	157C						
AF	1580	1E4						
AR	1768	17C						
QR	18E8	1DE						
QR	1AC8	1DE						
F1	1CA8	26A						
F2	1F18	208						
F3	2127	3CA						
F4	24F0	272						
F5	26F8	178						
F6	28D0	108						
F7	2AA8	176						
F8	2C20	176						
F9	2D98	184						
LINEAR	2F20	1FA						
SGN	3120	15A						
KUTTA	3280	282						
P8F	3538	1E2						
DGG4	3720	24A						
KUTTA1	3970	51C						
KUTTA2	3E90	3F4						
INHFEIT*	4288	1C						
INHNAMEL*	42A8	AAF	EXIT	4288				
INHNECOMH*	4D58	F8C	FRNL#	42A8	FWRNL#	4898		
INHNECOMH2*	5CE8	715	IBCOM#	4D84	FDIOCS#	4E40	INTSWCH	5C8E
INHSSCN *	6400	208	SEODASD	6116				
INHSSQRT*	6608	168	COS	6400	SIN	6422		
INHFCVTH*	6770	R6F	SQRT	6608				
INHNEFNTH*	72E0	548	ADCON#	6770	FCVAOUTP	681A	FCVLOUTP	68AA
INHNEFIOS*	7828	FF8	FCVIOUPT	6DAE	FCVEOUTP	6E40	FCVCOUTP	6FEA
INHNEFIOS2*	8820	588	ARITH#	72E0	ADJSWCH	767C		
INHNEARM *	8008	5FC	FIOCS#	7828	FIOCSBEP	782E		
			ERRNOM	80D8	IMNERRE			8DF0
							FCVZOUTP	6A06
							INT6SWCH	7260

NAME	ORIGIN	LENGTH	NAME	LOCATION	NAME	LOCATION	NAME	LOCATION	NAME	LOCATION
IHNNUOPT *	9308	318								
IHNFCONI *	96F0	2E5	FOCONI#	96F0						
IHNFCONO *	9908	442	FOCONO#	9908						
IHNETRCH *	9E80	246	IHNTRCH	9E80	ERRTRA	9E88				
IHNUTATBL *	A128	638								
IHNFTEN *	A760	198	FTEN#	A760						
BLK1	A8F8	8								
BLK2	A900	24								
BLK3	A928	10								
BLK4	A938	48								
BLK5	A980	28								

LOCATION	REFERS TO SYMBOL	IN CONTROL SECTION	LOCATION	REFERS TO SYMBOL	IN CONTROL SECTION
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388	BLK4		38C	BLK4	BLK4
39C	BLK4		388	BLK3	BLK3
3C8	BLK3		3D8	BLK3	BLK3
198	F7		19C	F8	F8
1A0	F9		1A4	IRCOM#	IRNECOMH
1A8	FRDNL#		1AC	FWNL#	IHNNAMEL
1R0	LINEAR		1R4	KUTTA	KUTTA
1R8	F1		18C	KUTTA2	KUTTA2
1C0	F5		70C	BLK5	BLK5
71C	BLK5		72C	BLK5	BLK5
73C	BLK5		74C	BLK4	BLK4
75C	BLK4		76C	BLK4	BLK4
77C	BLK4		78C	BLK4	BLK4
79C	BLK4		7AC	BLK4	BLK4
79C	BLK4		7CC	BLK4	BLK4
7DC	BLK4		7EC	BLK4	BLK4
7FC	BLK4		80C	BLK4	BLK4
81C	BLK4		82C	BLK4	BLK4
84C	BLK4		78	BLK1	BLK1
7C	BLK2		80	BLK3	BLK3
84	BLK4		88	BLK5	BLK5
1EC	BLK3		1FC	BLK3	BLK3
20C	BLK3		1C4	F6	F6
1C8	KUTTA1		1CC	F2	F2
1D0	F3		1D4	F4	F4
1D8	EXIT		1DC	SQRT	IHNSSQRT
6FC	BLK5		15FC	BLK5	BLK5
15F8	BLK4		17E4	BLK5	BLK5
17E0	BLK4		197C	AF	AF
1980	SGN		1984	SQRT	IHNSSQRT
1960	BLK4		185C	AR	AR
1860	SGN		1864	SQRT	IHNSSQRT

LOCATION	REFERS TO SYMBOL	IN CONTROL SECTION	LOCATION	REFERS TO SYMBOL	IN CONTROL SECTION
1840	BLK4	BLK4	1024	BLK4	BLK4
1028	RLK5	RLK5	1058	BLK4	BLK4
104C	PRF	PRF	1050	SIN	IHNSSCN
1020	RLK2	RLK2	256C	BLK4	BLK4
2570	RLK5	RLK5	258C	QR	QR
2590	AR	AR	2568	BLK2	BLK2
2770	RLK4	RLK4	2948	BLK4	BLK4
1F94	BLK4	BLK4	1F98	BLK5	BLK5
1FR4	QF	QF	1F88	AF	AF
1F90	BLK2	BLK2	219C	BLK4	BLK4
21A0	RLK5	RLK5	21C4	QF	QF
21C8	QR	QR	21CC	AR	AR
2100	AF	AF	2198	BLK2	BLK2
3314	F1	F1	32F8	BLK1	BLK1
3580	RLK3	RLK3	3A04	F2	F2
3A08	F3	F3	3A0C	F4	F4
39E8	BLK1	BLK1	3F24	F5	F5
3F28	F6	F6	3F08	BLK1	BLK1
42A0	IRCOM#	IRNECOMH	4828	IBCOM#	IRNECOMH
4830	ADCON#	IRNEFCVTH	482C	FIOCS#	IRNEFIO5
4C14	IHNERRM	IHNERRM	4E40	SEQDASD	IRNECOMH2
4E94	IRNECOMH2	IRNECOMH2	58C4	ADCON#	IRNEFCVTH
588C	FIOCS#	IRNEFIO5	58C8	ARITH#	IRNEFNTH
58E8	ADJSWICH	IRNEFNTH	58E4	IHNUOPT	IHNUOPT
5CE0	IHNCMSE	\$UNRESOLVED(W)	58CC	FCVEOUTP	IRNEFCVTH
58D0	FCVLOUTP	IRNEFCVTH	58D4	FCVIOUPT	IRNEFCVTH
58D8	FCVCOUTP	IRNEFCVTH	58DC	FCVAOUTP	IRNEFCVTH
58E0	FCVZOUTP	IRNEFCVTH	5870	IHNERR	IHNERRM
589C	IRNECOMH2	IRNECOMH2	58A0	IHNERRM	IHNERRM
5874	IRNECOMH2	IRNECOMH2	5878	IRNECOMH2	IRNECOMH2
587C	IRNECOMH2	IRNECOMH2	58A0	IRNECOMH2	IRNECOMH2
5F80	IRNECOMH	IRNECOMH	5FD9	IRNECOMH	IRNECOMH
5FDC	IRNECOMH	IRNECOMH	5D30	IHNERRM	IRNECOMH
5D2C	IRCOM#	IRNECOMH	6250	IRNECOMH	IRNECOMH
626D	IRNECOMH	IRNECOMH	627D	IRNECOMH	IRNECOMH
6550	IRCOM#	IRNECOMH	6594	IHNERRM	IHNERRM
66EC	IRCOM#	IRNECOMH	6714	IHNERRM	IHNERRM
70D0	IRCOM#	IRNECOMH	70CC	IHNERRM	IHNERRM
7120	FQCONO#	IRNECOMH	7124	FQCONI#	IRNECONI
76CC	IRCOM#	IRNECOMH	76D0	INTSWICH	IRNECOMH
7678	INT6SWCH	IRNEFCVTH	7674	IHNUOPT	IHNUOPT
76D8	ADCON#	IRNEFCVTH	7604	FIOCS#	IRNEFIO5
7744	IHNERRM	IHNERRM	7988	IHNERRM	IHNERRM
798C	IRNEFIO52	IRNEFIO52	862C	IHNUATBL	IRNEFIO52
8638	IRCOM#	IRNECOMH	864D	IRNEFIO52	IRNEFIO52
8664	IRNEFIO52	IRNEFIO52	8819	IRNEFIO52	IRNEFIO52
93C4	IHNUOPT	IHNUOPT	93C8	IRCOM#	IRNECOMH
93CC	IHNTRCH	IHNTRCH	93D0	FIOCSREP	IRNEFIO5
996C	FTEN#	IRNEFIO5	90C8	FTEN#	IRNEFIO5
A010	LDFIO#	\$UNRESOLVED(W)	A008	IRCOM#	IRNECOMH
A00C	ADCON#	IRNEFCVTH	A014	FIOCSBEP	IRNEFIO5

ENTRY ADDRESS 00
TOTAL LENGTH A9AB

T=0.0339 X=21.188 XD= 69.54 XDD= 1975.4 P1= 2865.88 PID= 0.0 P2= 2865.88 P2D= -7733.6 P3= 2865.88 P3D= 0.0
 DELA= 0.0188 DELAD=-0.0484 DELD=-0.0053 DELDD= 0.0121 SIGT= 55205.4 SIGAX= 26169.8 SIGR= -2865.9 SIGE= -2865.9 SIGE= 35561.2
 T=0.0359 X=21.330 XD= 73.48 XDD= 1962.9 P1= 2849.89 PID= 0.0 P2= 2849.89 P2D= -8276.5 P3= 2849.89 P3D= 0.0
 DELA= 0.0187 DELAD=-0.0490 DELD=-0.0053 DELDD= 0.0121 SIGT= 54897.3 SIGAX= 26023.7 SIGR= -2849.9 SIGE= -2849.9 SIGE= 35362.8
 T=0.0379 X=21.481 XD= 77.39 XDD= 1949.7 P1= 2833.04 PID= 0.0 P2= 2833.04 P2D= -8594.9 P3= 2833.04 P3D= 0.0
 DELA= 0.0186 DELAD=-0.0537 DELD=-0.0053 DELDD= 0.0135 SIGT= 54572.7 SIGAX= 25869.9 SIGR= -2833.0 SIGE= -2833.0 SIGE= 35153.7
 T=0.0399 X=21.634 XD= 81.29 XDD= 1935.8 P1= 2815.33 PID= 0.0 P2= 2815.33 P2D= -9022.3 P3= 2815.33 P3D= 0.0
 DELA= 0.0185 DELAD=-0.0564 DELD=-0.0052 DELDD= 0.0139 SIGT= 54231.6 SIGAX= 25708.1 SIGR= -2815.3 SIGE= -2815.3 SIGE= 34933.9
 T=0.0419 X=21.805 XD= 85.13 XDD= 1921.3 P1= 2796.78 PID= 0.0 P2= 2796.78 P2D= -9500.4 P3= 2796.78 P3D= 0.0
 DELA= 0.0184 DELAD=-0.0576 DELD=-0.0052 DELDD= 0.0149 SIGT= 53874.2 SIGAX= 25538.8 SIGR= -2796.8 SIGE= -2796.8 SIGE= 34703.7
 T=0.0439 X=21.979 XD= 88.96 XDD= 1906.1 P1= 2777.40 PID= 0.0 P2= 2777.40 P2D= -9923.0 P3= 2777.40 P3D= 0.0
 DELA= 0.0182 DELAD=-0.0601 DELD=-0.0052 DELDD= 0.0154 SIGT= 53500.9 SIGAX= 25361.8 SIGR= -2777.4 SIGE= -2777.4 SIGE= 34463.3
 T=0.0459 X=22.161 XD= 92.76 XDD= 1890.3 P1= 2757.19 PID= 0.0 P2= 2757.19 P2D= -10235.4 P3= 2757.19 P3D= 0.0
 DELA= 0.0181 DELAD=-0.0647 DELD=-0.0051 DELDD= 0.0156 SIGT= 53111.7 SIGAX= 25177.3 SIGR= -2757.2 SIGE= -2757.2 SIGE= 34212.5
 T=0.0479 X=22.350 XD= 96.52 XDD= 1873.8 P1= 2736.16 PID= 0.0 P2= 2736.16 P2D= -10696.6 P3= 2736.16 P3D= 0.0
 DELA= 0.0180 DELAD=-0.0660 DELD=-0.0051 DELDD= 0.0175 SIGT= 52706.6 SIGAX= 24985.2 SIGR= -2736.2 SIGE= -2736.2 SIGE= 33951.6
 T=0.0499 X=22.546 XD= 100.25 XDD= 1856.7 P1= 2714.34 PID= 0.0 P2= 2714.34 P2D= -11183.4 P3= 2714.34 P3D= 0.0
 DELA= 0.0178 DELAD=-0.0665 DELD=-0.0051 DELDD= 0.0165 SIGT= 52286.2 SIGAX= 24786.0 SIGR= -2714.3 SIGE= -2714.3 SIGE= 33680.8
 T=0.0519 X=22.750 XD= 103.95 XDD= 1839.0 P1= 2691.72 PID= 0.0 P2= 2691.72 P2D= -11465.9 P3= 2691.72 P3D= 0.0
 DELA= 0.0177 DELAD=-0.0712 DELD=-0.0050 DELDD= 0.0179 SIGT= 51850.6 SIGAX= 24579.4 SIGR= -2691.7 SIGE= -2691.7 SIGE= 33400.2
 T=0.0539 X=22.961 XD= 107.60 XDD= 1820.6 P1= 2668.32 PID= 0.0 P2= 2668.32 P2D= -11841.9 P3= 2668.32 P3D= 0.0
 DELA= 0.0175 DELAD=-0.0741 DELD=-0.0050 DELDD= 0.0181 SIGT= 51399.7 SIGAX= 24365.7 SIGR= -2668.3 SIGE= -2668.3 SIGE= 33109.7
 T=0.0559 X=23.180 XD= 111.23 XDD= 1801.7 P1= 2644.14 PID= 0.0 P2= 2644.14 P2D= -12340.7 P3= 2644.14 P3D= 0.0
 DELA= 0.0174 DELAD=-0.0739 DELD=-0.0049 DELDD= 0.0191 SIGT= 50934.1 SIGAX= 24145.0 SIGR= -2644.1 SIGE= -2644.1 SIGE= 32809.8
 T=0.0579 X=23.405 XD= 114.81 XDD= 1782.1 P1= 2619.22 PID= 0.0 P2= 2619.22 P2D= -12661.7 P3= 2619.22 P3D= 0.0
 DELA= 0.0172 DELAD=-0.0772 DELD=-0.0049 DELDD= 0.0198 SIGT= 50454.0 SIGAX= 23917.4 SIGR= -2619.2 SIGE= -2619.2 SIGE= 32500.5
 T=0.0599 X=23.638 XD= 118.35 XDD= 1762.0 P1= 2593.54 PID= 0.0 P2= 2593.54 P2D= -12956.3 P3= 2593.54 P3D= 0.0
 DELA= 0.0170 DELAD=-0.0811 DELD=-0.0048 DELDD= 0.0203 SIGT= 49959.3 SIGAX= 23682.9 SIGR= -2593.5 SIGE= -2593.5 SIGE= 32181.9
 T=0.0619 X=23.878 XD= 121.85 XDD= 1741.3 P1= 2567.13 PID= 0.0 P2= 2567.13 P2D= -13380.0 P3= 2567.13 P3D= 0.0
 DELA= 0.0169 DELAD=-0.0821 DELD=-0.0048 DELDD= 0.0209 SIGT= 49450.5 SIGAX= 23441.7 SIGR= -2567.1 SIGE= -2567.1 SIGE= 31854.2
 T=0.0639 X=24.125 XD= 125.32 XDD= 1720.0 P1= 2540.00 PID= 0.0 P2= 2540.00 P2D= -13757.2 P3= 2540.00 P3D= 0.0
 DELA= 0.0167 DELAD=-0.0836 DELD=-0.0047 DELDD= 0.0216 SIGT= 48927.9 SIGAX= 23194.0 SIGR= -2540.0 SIGE= -2540.0 SIGE= 31517.5
 T=0.0659 X=24.379 XD= 128.73 XDD= 1698.2 P1= 2512.16 PID= 0.0 P2= 2512.16 P2D= -14074.1 P3= 2512.16 P3D= 0.0
 DELA= 0.0165 DELAD=-0.0870 DELD=-0.0047 DELDD= 0.0215 SIGT= 48391.7 SIGAX= 22939.8 SIGR= -2512.2 SIGE= -2512.2 SIGE= 31172.1
 T=0.0679 X=24.639 XD= 132.11 XDD= 1675.8 P1= 2483.62 PID= 0.0 P2= 2483.62 P2D= -14433.8 P3= 2483.62 P3D= 0.0
 DELA= 0.0163 DELAD=-0.0887 DELD=-0.0046 DELDD= 0.0233 SIGT= 47842.0 SIGAX= 22679.2 SIGR= -2483.6 SIGE= -2483.6 SIGE= 30818.0
 T=0.0699 X=24.907 XD= 135.43 XDD= 1652.9 P1= 2454.41 PID= 0.0 P2= 2454.41 P2D= -14804.4 P3= 2454.41 P3D= 0.0
 DELA= 0.0161 DELAD=-0.0894 DELD=-0.0046 DELDD= 0.0233 SIGT= 47279.2 SIGAX= 22412.4 SIGR= -2454.4 SIGE= -2454.4 SIGE= 30455.5
 T=0.0719 X=25.180 XD= 138.72 XDD= 1629.5 P1= 2424.53 PID= 0.0 P2= 2424.53 P2D= -15077.2 P3= 2424.53 P3D= 0.0
 DELA= 0.0159 DELAD=-0.0933 DELD=-0.0045 DELDD= 0.0239 SIGT= 46703.6 SIGAX= 22139.6 SIGR= -2424.5 SIGE= -2424.5 SIGE= 30084.7
 T=0.0739 X=25.461 XD= 141.95 XDD= 1605.6 P1= 2393.99 PID= 0.0 P2= 2393.99 P2D= -15414.6 P3= 2393.99 P3D= 0.0
 DELA= 0.0157 DELAD=-0.0949 DELD=-0.0045 DELDD= 0.0239 SIGT= 46115.4 SIGAX= 21860.7 SIGR= -2394.0 SIGE= -2394.0 SIGE= 29705.8
 T=0.0759 X=25.748 XD= 145.14 XDD= 1581.1 P1= 2362.82 PID= 0.0 P2= 2362.82 P2D= -15772.3 P3= 2362.82 P3D= 0.0
 DELA= 0.0155 DELAD=-0.0958 DELD=-0.0044 DELDD= 0.0247 SIGT= 45514.9 SIGAX= 21576.0 SIGR= -2362.8 SIGE= -2362.8 SIGE= 29319.0
 T=0.0779 X=26.041 XD= 148.27 XDD= 1556.2 P1= 2331.02 PID= 0.0 P2= 2331.02 P2D= -16011.8 P3= 2331.02 P3D= 0.0
 DELA= 0.0153 DELAD=-0.0991 DELD=-0.0043 DELDD= 0.0252 SIGT= 44902.3 SIGAX= 21285.7 SIGR= -2331.0 SIGE= -2331.0 SIGE= 28924.4
 T=0.0799 X=26.340 XD= 151.36 XDD= 1530.8 P1= 2298.61 PID= 0.0 P2= 2298.61 P2D= -16330.5 P3= 2298.61 P3D= 0.0
 DELA= 0.0151 DELAD=-0.1007 DELD=-0.0043 DELDD= 0.0255 SIGT= 44278.0 SIGAX= 20989.7 SIGR= -2298.6 SIGE= -2298.6 SIGE= 28522.2
 T=0.0819 X=26.646 XD= 154.39 XDD= 1504.9 P1= 2265.60 PID= 0.0 P2= 2265.60 P2D= -16675.6 P3= 2265.60 P3D= 0.0
 DELA= 0.0149 DELAD=-0.1017 DELD=-0.0042 DELDD= 0.0256 SIGT= 43642.2 SIGAX= 20688.3 SIGR= -2265.6 SIGE= -2265.6 SIGE= 28112.7
 T=0.0839 X=26.957 XD= 157.38 XDD= 1478.6 P1= 2232.02 PID= 0.0 P2= 2232.02 P2D= -16907.9 P3= 2232.02 P3D= 0.0
 DELA= 0.0147 DELAD=-0.1043 DELD=-0.0042 DELDD= 0.0264 SIGT= 42995.3 SIGAX= 20381.7 SIGR= -2232.0 SIGE= -2232.0 SIGE= 27696.0
 T=0.0859 X=27.274 XD= 160.31 XDD= 1451.8 P1= 2197.87 PID= 0.0 P2= 2197.87 P2D= -17198.1 P3= 2197.87 P3D= 0.0
 DELA= 0.0144 DELAD=-0.1057 DELD=-0.0041 DELDD= 0.0270 SIGT= 42337.5 SIGAX= 20069.8 SIGR= -2197.9 SIGE= -2197.9 SIGE= 27272.2
 T=0.0879 X=27.598 XD= 163.18 XDD= 1424.6 P1= 2163.17 PID= 0.0 P2= 2163.17 P2D= -17497.7 P3= 2163.17 P3D= 0.0
 DELA= 0.0142 DELAD=-0.1070 DELD=-0.0040 DELDD= 0.0271 SIGT= 41664.2 SIGAX= 19753.0 SIGR= -2163.2 SIGE= -2163.2 SIGE= 26841.7
 T=0.0899 X=27.926 XD= 166.00 XDD= 1397.0 P1= 2127.95 PID= 0.0 P2= 2127.95 P2D= -17754.4 P3= 2127.95 P3D= 0.0
 DELA= 0.0140 DELAD=-0.1088 DELD=-0.0040 DELDD= 0.0274 SIGT= 40990.5 SIGAX= 19431.3 SIGR= -2127.9 SIGE= -2127.9 SIGE= 26404.6
 T=0.0919 X=28.261 XD= 168.77 XDD= 1368.9 P1= 2092.20 PID= 0.0 P2= 2092.20 P2D= -17978.0 P3= 2092.20 P3D= 0.0
 DELA= 0.0137 DELAD=-0.1107 DELD=-0.0039 DELDD= 0.0282 SIGT= 40301.9 SIGAX= 19104.9 SIGR= -2092.2 SIGE= -2092.2 SIGE= 25961.0
 T=0.0939 X=28.601 XD= 171.47 XDD= 1340.5 P1= 2055.95 PID= 0.0 P2= 2055.95 P2D= -18259.8 P3= 2055.95 P3D= 0.0
 DELA= 0.0135 DELAD=-0.1115 DELD=-0.0038 DELDD= 0.0286 SIGT= 39603.7 SIGAX= 18773.9 SIGR= -2055.9 SIGE= -2055.9 SIGE= 25511.2
 T=0.0959 X=28.946 XD= 174.13 XDD= 1311.7 P1= 2019.22 PID= 0.0 P2= 2019.22 P2D= -18468.6 P3= 2019.22 P3D= 0.0
 DELA= 0.0133 DELAD=-0.1137 DELD=-0.0038 DELDD= 0.0288 SIGT= 38996.1 SIGAX= 18438.5 SIGR= -2019.2 SIGE= -2019.2 SIGE= 25055.4
 T=0.0979 X=29.297 XD= 176.72 XDD= 1282.5 P1= 1982.02 PID= 0.0 P2= 1982.02 P2D= -18701.0 P3= 1982.02 P3D= 0.0

DELA= 0.0130 DELAD=-0.1151 DELD=-0.0037 DELDD=0.0292 SIGT= 38179.5 SIGAX= 18098.8 SIGR= -1982.0 SIGE= 24593.8
 T=0.0999 X=29.652 XD= 179.25 XDD= 1253.0 P1= 1944.36 P1D= 1944.36 P2D= -18957.8 P3= 1944.36 P3D= 0.0
 DELA= 0.0128 DELAD=-0.1157 DELD=-0.0036 DELDD=0.0296 SIGT= 37454.2 SIGAX= 17754.9 SIGR= -1944.4 SIGE= 24126.6
 T=0.1019 X=30.013 XD= 181.73 XDD= 1223.1 P1= 1906.27 P1D= 1906.27 P2D= -19123.1 P3= 1906.27 P3D= 0.0
 DELA= 0.0125 DELAD=-0.1180 DELD=-0.0035 DELDD=0.0300 SIGT= 36720.4 SIGAX= 17407.1 SIGR= -1906.3 SIGE= 23653.9
 T=0.1039 X=30.379 XD= 184.14 XDD= 1192.9 P1= 1867.76 P1D= 1867.76 P2D= -19348.8 P3= 1867.76 P3D= 0.0
 DELA= 0.0123 DELAD=-0.1190 DELD=-0.0035 DELDD=0.0301 SIGT= 35978.5 SIGAX= 16775.4 SIGR= -1867.8 SIGE= 23176.0
 T=0.1059 X=30.749 XD= 186.50 XDD= 1162.4 P1= 1828.94 P1D= 1828.94 P2D= -19586.8 P3= 1828.94 P3D= 0.0
 DELA= 0.0120 DELAD=-0.1192 DELD=-0.0034 DELDD=0.0306 SIGT= 35228.9 SIGAX= 16700.0 SIGR= -1828.8 SIGE= 22893.1
 T=0.1079 X=31.124 XD= 184.79 XDD= 1131.6 P1= 1789.54 P1D= 1789.54 P2D= -19712.9 P3= 1789.54 P3D= 0.0
 DELA= 0.0118 DELAD=-0.1218 DELD=-0.0033 DELDD=0.0309 SIGT= 34471.8 SIGAX= 16341.1 SIGR= -1789.5 SIGE= 22205.4
 T=0.1099 X=31.504 XD= 191.02 XDD= 1100.5 P1= 1749.86 P1D= 1749.86 P2D= -19911.2 P3= 1749.86 P3D= 0.0
 DELA= 0.0115 DELAD=-0.1225 DELD=-0.0033 DELDD=0.0311 SIGT= 33707.5 SIGAX= 15978.8 SIGR= -1749.9 SIGE= 21713.1
 T=0.1119 X=31.887 XD= 193.19 XDD= 1069.1 P1= 1709.83 P1D= 1709.83 P2D= -20130.1 P3= 1709.83 P3D= 0.0
 DELA= 0.0112 DELAD=-0.1226 DELD=-0.0032 DELDD=0.0315 SIGT= 32936.4 SIGAX= 15613.3 SIGR= -1709.8 SIGE= 21216.4
 T=0.1139 X=32.276 XD= 195.30 XDD= 1037.4 P1= 1669.46 P1D= 1669.46 P2D= -20228.1 P3= 1669.46 P3D= 0.0
 DELA= 0.0110 DELAD=-0.1258 DELD=-0.0031 DELDD=0.0315 SIGT= 32158.8 SIGAX= 15244.7 SIGR= -1669.5 SIGE= 20715.5
 T=0.1159 X=32.668 XD= 197.34 XDD= 1005.5 P1= 1628.77 P1D= 1628.77 P2D= -20426.4 P3= 1628.77 P3D= 0.0
 DELA= 0.0107 DELAD=-0.1251 DELD=-0.0030 DELDD=0.0320 SIGT= 31375.0 SIGAX= 14873.1 SIGR= -1628.8 SIGE= 20210.6
 T=0.1179 X=33.064 XD= 199.32 XDD= 973.4 P1= 1587.78 P1D= 1587.78 P2D= -20583.6 P3= 1587.78 P3D= 0.0
 DELA= 0.0104 DELAD=-0.1276 DELD=-0.0028 DELDD=0.0325 SIGT= 29790.1 SIGAX= 14498.8 SIGR= -1587.8 SIGE= 19701.9
 T=0.1199 X=33.465 XD= 201.23 XDD= 941.0 P1= 1546.49 P1D= 1546.49 P2D= -20687.9 P3= 1546.49 P3D= 0.0
 DELA= 0.0102 DELAD=-0.1275 DELD=-0.0029 DELDD=0.0323 SIGT= 28989.6 SIGAX= 1421.8 SIGR= -1546.5 SIGE= 19189.6
 T=0.1219 X=33.869 XD= 203.08 XDD= 908.4 P1= 1504.94 P1D= 1504.94 P2D= -20853.5 P3= 1504.94 P3D= 0.0
 DELA= 0.0099 DELAD=-0.1276 DELD=-0.0028 DELDD=0.0325 SIGT= 28184.2 SIGAX= 13742.3 SIGR= -1504.9 SIGE= 18674.0
 T=0.1239 X=34.276 XD= 204.86 XDD= 875.6 P1= 1463.13 P1D= 1463.13 P2D= -20948.3 P3= 1463.13 P3D= 0.0
 DELA= 0.0096 DELAD=-0.1290 DELD=-0.0027 DELDD=0.0329 SIGT= 27374.2 SIGAX= 13360.6 SIGR= -1463.1 SIGE= 18155.2
 T=0.1259 X=34.687 XD= 206.58 XDD= 842.6 P1= 1421.08 P1D= 1421.08 P2D= -21064.5 P3= 1421.08 P3D= 0.0
 DELA= 0.0093 DELAD=-0.1297 DELD=-0.0026 DELDD=0.0332 SIGT= 26559.9 SIGAX= 12974.5 SIGR= -1421.1 SIGE= 17633.4
 T=0.1279 X=35.102 XD= 208.23 XDD= 809.5 P1= 1378.80 P1D= 1378.80 P2D= -21214.8 P3= 1378.80 P3D= 0.0
 DELA= 0.0091 DELAD=-0.1294 DELD=-0.0026 DELDD=0.0332 SIGT= 25741.6 SIGAX= 12590.5 SIGR= -1378.8 SIGE= 17108.9
 T=0.1299 X=35.520 XD= 209.81 XDD= 776.1 P1= 1336.33 P1D= 1336.33 P2D= -21246.6 P3= 1336.33 P3D= 0.0
 DELA= 0.0088 DELAD=-0.1315 DELD=-0.0025 DELDD=0.0332 SIGT= 25741.6 SIGAX= 12202.6 SIGR= -1336.3 SIGE= 16581.8
 T=0.1319 X=35.941 XD= 211.33 XDD= 742.7 P1= 1293.65 P1D= 1293.65 P2D= -21384.3 P3= 1293.65 P3D= 0.0
 DELA= 0.0085 DELAD=-0.1310 DELD=-0.0024 DELDD=0.0334 SIGT= 24919.6 SIGAX= 11813.0 SIGR= -1293.7 SIGE= 16052.3
 T=0.1339 X=36.364 XD= 212.78 XDD= 709.1 P1= 1250.81 P1D= 1250.81 P2D= -21459.0 P3= 1250.81 P3D= 0.0
 DELA= 0.0082 DELAD=-0.1318 DELD=-0.0023 DELDD=0.0335 SIGT= 24094.3 SIGAX= 11421.8 SIGR= -1250.8 SIGE= 15520.7
 T=0.1359 X=36.791 XD= 214.16 XDD= 675.3 P1= 1207.81 P1D= 1207.81 P2D= -21517.4 P3= 1207.81 P3D= 0.0
 DELA= 0.0079 DELAD=-0.1326 DELD=-0.0022 DELDD=0.0336 SIGT= 23265.9 SIGAX= 11029.1 SIGR= -1207.8 SIGE= 14987.0
 T=0.1379 X=37.220 XD= 215.48 XDD= 641.5 P1= 1164.66 P1D= 1164.66 P2D= -21629.3 P3= 1164.66 P3D= 0.0
 DELA= 0.0077 DELAD=-0.1320 DELD=-0.0022 DELDD=0.0334 SIGT= 22434.8 SIGAX= 10635.1 SIGR= -1164.7 SIGE= 14451.7
 T=0.1399 X=37.652 XD= 216.73 XDD= 607.6 P1= 1121.39 P1D= 1121.39 P2D= -21626.6 P3= 1121.39 P3D= 0.0
 DELA= 0.0074 DELAD=-0.1338 DELD=-0.0021 DELDD=0.0339 SIGT= 19928.1 SIGAX= 10240.0 SIGR= -1121.4 SIGE= 13914.7
 T=0.1419 X=38.087 XD= 217.91 XDD= 573.5 P1= 1078.00 P1D= 1078.00 P2D= -21737.5 P3= 1078.00 P3D= 0.0
 DELA= 0.0071 DELAD=-0.1327 DELD=-0.0020 DELDD=0.0340 SIGT= 20765.6 SIGAX= 9843.8 SIGR= -1078.0 SIGE= 13376.4
 T=0.1439 X=38.523 XD= 219.02 XDD= 539.4 P1= 1034.53 P1D= 1034.53 P2D= -21752.8 P3= 1034.53 P3D= 0.0
 DELA= 0.0068 DELAD=-0.1337 DELD=-0.0019 DELDD=0.0339 SIGT= 19928.1 SIGAX= 9446.8 SIGR= -1034.5 SIGE= 12836.9
 T=0.1459 X=38.962 XD= 220.06 XDD= 505.3 P1= 990.97 P1D= 990.97 P2D= -21789.1 P3= 990.97 P3D= 0.0
 DELA= 0.0065 DELAD=-0.1338 DELD=-0.0018 DELDD=0.0341 SIGT= 19089.0 SIGAX= 9049.0 SIGR= -991.0 SIGE= 12296.4
 T=0.1479 X=39.403 XD= 221.04 XDD= 471.0 P1= 947.35 P1D= 947.35 P2D= -21830.9 P3= 947.35 P3D= 0.0
 DELA= 0.0062 DELAD=-0.1337 DELD=-0.0018 DELDD=0.0341 SIGT= 18248.7 SIGAX= 8650.7 SIGR= -947.3 SIGE= 11755.1
 T=0.1499 X=39.846 XD= 221.95 XDD= 436.8 P1= 903.67 P1D= 903.67 P2D= -21827.6 P3= 903.67 P3D= 0.0
 DELA= 0.0059 DELAD=-0.1345 DELD=-0.0017 DELDD=0.0341 SIGT= 17407.4 SIGAX= 8251.9 SIGR= -903.7 SIGE= 11213.2
 T=0.1519 X=40.290 XD= 222.77 XDD= 379.4 P1= 889.47 P1D= 889.47 P2D= -21872.0 P3= 889.47 P3D= 0.0
 DELA= 0.0055 DELAD=-0.1345 DELD=-0.0017 DELDD=0.0341 SIGT= 16208.1 SIGAX= 7829.28 SIGR= -889.47 P1D= 889.47 P2D= -21872.0 P3= 889.47 P3D= 0.0
 DELA= 0.0052 DELAD=-0.1345 DELD=-0.0017 DELDD=0.0341 SIGT= 15974.4 SIGAX= 7572.6 SIGR= -829.3 SIGE= 10290.1
 T=0.1539 X=40.736 XD= 223.45 XDD= 308.8 P1= 889.47 P1D= 889.47 P2D= -21872.0 P3= 889.47 P3D= 0.0
 DELA= 0.0049 DELAD=-0.1345 DELD=-0.0017 DELDD=0.0341 SIGT= 14239.7 SIGAX= 739.23 SIGR= -28371.3 P3= 739.23 P3D= 0.0
 T=0.1559 X=41.183 XD= 224.00 XDD= 237.4 P1= 889.47 P1D= 889.47 P2D= -21872.0 P3= 889.47 P3D= 0.0
 DELA= 0.0043 DELAD=-0.1384 DELD=-0.0012 DELDD=0.0606 SIGT= 12494.2 SIGAX= 6848.61 SIGR= -13475.3 P3= 6848.61 P3D= 0.0
 T=0.1578 X=41.631 XD= 224.40 XDD= 165.9 P1= 889.47 P1D= 889.47 P2D= -21872.0 P3= 889.47 P3D= 0.0
 DELA= 0.0037 DELAD=-0.1496 DELD=-0.0010 DELDD=0.0851 SIGT= 10753.7 SIGAX= 5582.6 SIGR= -17634.1 P3= 5582.6 P3D= 0.0
 T=0.1598 X=42.080 XD= 224.65 XDD= 96.2 P1= 889.47 P1D= 889.47 P2D= -21872.0 P3= 889.47 P3D= 0.0
 DELA= 0.0031 DELAD=-0.1384 DELD=-0.0009 DELDD=0.0564 SIGT= 9025.3 SIGAX= 5097.7 SIGR= -558.3 SIGE= 8048.3
 T=0.1618 X=42.529 XD= 224.78 XDD= 25.7 P1= 889.47 P1D= 889.47 P2D= -21872.0 P3= 889.47 P3D= 0.0
 DELA= 0.0025 DELAD=-0.2067 DELD=-0.0007 DELDD=0.0787 SIGT= 7287.8 SIGAX= 468.53 SIGR= -3577.8 P3= 468.53 P3D= 0.0
 DELA= 0.0025 DELAD=-0.2067 DELD=-0.0007 DELDD=0.0787 SIGT= 7287.8 SIGAX= 378.33 SIGR= -468.5 SIGE= 5813.7
 DELA= 0.0025 DELAD=-0.2067 DELD=-0.0007 DELDD=0.0787 SIGT= 7287.8 SIGAX= 3454.7 SIGR= -378.3 SIGE= 4694.5

T=0.1638 X=2.974 XD= 215.26 XDD= -12757.3 P1= A99.47 PID= -617611.4 P2= DELA= 0.0019 DELAD= -0.2839 DELD= -0.0005 DELDD= 0.0633 SIGT= 5594.6 SIGX= 0.0
T=0.1658 X=3.357 XD= 153.25 XDD= -57045.0 P1= A99.47 PID= -121377.0 P2= DELA= 0.0014 DELAD= -0.1987 DELD= -0.0004 DELDD= 0.0439 SIGT= 4151.4 SIGX= 0.0
T=0.1678 X=3.510 XD= -14.75 XDD= -98547.5 P1= A99.47 PID= -2606387.0 P2= DELA= 0.0012 DELAD= 0.0585 DELD= -0.0004 DELDD= 0.0000 SIGT= 3635.0 SIGX= 0.0
T=0.1698 X=3.298 XD= -185.17 XDD= -66502.7 P1= A99.47 PID= -3885951.0 P2= DELA= 0.0015 DELAD= 0.1551 DELD= -0.0004 DELDD= -0.0635 SIGT= 4403.3 SIGX= 0.0
T=0.1718 X=4.218 XD= -284.10 XDD= -35582.8 P1= A99.47 PID= -4417347.0 P2= DELA= 0.0021 DELAD= 0.4326 DELD= -0.0006 DELDD= -0.0808 SIGT= 6258.1 SIGX= 0.0
T=0.1738 X=4.219 XD= -337.69 XDD= -19932.7 P1= A99.47 PID= -4455623.0 P2= DELA= 0.0030 DELAD= 0.3523 DELD= -0.0008 DELDD= -0.1185 SIGT= 8743.7 SIGX= 0.0
T=0.1758 X=4.148 XD= -371.01 XDD= -14870.1 P1= A99.47 PID= -4218163.0 P2= DELA= 0.0039 DELAD= 0.6354 DELD= -0.0011 DELDD= -0.1144 SIGT= 11595.2 SIGX= 0.0
T=0.1778 X=4.0710 XD= -397.93 XDD= -12173.9 P1= A99.47 PID= -3742950.0 P2= DELA= 0.0050 DELAD= 0.4992 DELD= -0.0014 DELDD= -0.1352 SIGT= 14739.0 SIGX= 0.0
T=0.1798 X=3.892 XD= -419.90 XDD= -9909.6 P1= A99.47 PID= 17887.6 SIGX= 0.0
T=0.1818 X=3.833 XD= -437.83 XDD= -8105.2 P1= 1012.89 PID= 19511.3 SIGX= 0.0
T=0.1838 X=3.8142 XD= -452.45 XDD= -8581.3 P1= 1101.76 PID= 21223.1 SIGX= 0.0
T=0.1858 X=3.7224 XD= -464.26 XDD= -5290.7 P1= 1193.92 PID= 22998.5 SIGX= 0.0
T=0.1878 X=3.6286 XD= -473.69 XDD= -4188.8 P1= 1287.74 PID= 24805.7 SIGX= 0.0
T=0.1898 X=3.5331 XD= -481.08 XDD= -3241.9 P1= 1395.42 PID= 26687.4 SIGX= 0.0
T=0.1918 X=3.4362 XD= -486.71 XDD= -2425.2 P1= 1483.82 PID= 28582.8 SIGX= 0.0
T=0.1938 X=3.3384 XD= -490.82 XDD= -1713.1 P1= 1584.21 PID= 30516.7 SIGX= 0.0
T=0.1958 X=3.2399 XD= -493.59 XDD= -1090.1 P1= 1686.68 PID= 32490.4 SIGX= 0.0
T=0.1978 X=3.1410 XD= -495.20 XDD= -541.2 P1= 1789.11 PID= 34463.6 SIGX= 0.0
T=0.1998 X=3.0419 XD= -495.77 XDD= -52.4 P1= 1893.28 PID= 36470.2 SIGX= 0.0
T=0.2018 X=2.9427 XD= -495.42 XDD= 383.7 P1= 1998.27 PID= 38492.7 SIGX= 0.0
T=0.2038 X=2.8437 XD= -494.24 XDD= 776.4 P1= 2102.97 PID= 40509.5 SIGX= 0.0
T=0.2058 X=2.7450 XD= -492.32 XDD= 1133.5 P1= 2208.80 PID= 42548.0 SIGX= 0.0
T=0.2078 X=2.6468 XD= -489.71 XDD= 1458.6 P1= 2314.83 PID= 44590.6 SIGX= 0.0
T=0.2098 X=2.5491 XD= -486.48 XDD= 1757.3 P1= 2420.26 PID= 46621.5 SIGX= 0.0
T=0.2118 X=2.4521 XD= -482.68 XDD= 2034.2 P1= 2526.16 PID= 48661.3 SIGX= 0.0
T=0.2138 X=2.3560 XD= -478.34 XDD= 2291.2 P1= 2631.99 PID= 50700.0 SIGX= 0.0
T=0.2158 X=2.2608 XD= -473.51 XDD= 2531.1 P1= 2736.91 PID= 52720.9 SIGX= 0.0
T=0.2178 X=2.1666 XD= -468.21 XDD= 2757.2 P1= 2841.55 PID= 54736.8 SIGX= 0.0
T=0.2198 X=2.0735 XD= -467.47 XDD= 2970.5 P1= 2945.99 PID= 56748.5 SIGX= 0.0
T=0.2218 X=1.9816 XD= -455.84 XDD= 5378.3 P1= 3014.76 PID= 58073.3 SIGX= 0.0
T=0.2238 X=1.8919 XD= -439.12 XDD= 10016.1 P1= 3066.78 PID= 59075.3 SIGX= 0.0
T=0.2258 X=1.8062 XD= -417.79 XDD= 11052.6 P1= 3157.68 PID= 60826.2 SIGX= 0.0
T=0.2278 X=1.7248 XD= -395.35 XDD= 11315.6 P1= 3251.29 PID= 62851.0 SIGX= 0.0
T=0.2298 X=1.6450 XD= -370.12 XDD= 11611.1 P1= 3351.29 PID= 64881.0 SIGX= 0.0
T=0.2318 X=1.5675 XD= -343.83 XDD= 11861.1 P1= 3451.29 PID= 66911.0 SIGX= 0.0
T=0.2338 X=1.4919 XD= -316.54 XDD= 12061.1 P1= 3551.29 PID= 68941.0 SIGX= 0.0
T=0.2358 X=1.4184 XD= -289.25 XDD= 12261.1 P1= 3651.29 PID= 70971.0 SIGX= 0.0
T=0.2378 X=1.3469 XD= -261.96 XDD= 12461.1 P1= 3751.29 PID= 73001.0 SIGX= 0.0
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T=0.2418 X=1.2099 XD= -207.38 XDD= 12861.1 P1= 3951.29 PID= 77061.0 SIGX= 0.0
T=0.2438 X=1.1444 XD= -180.09 XDD= 13061.1 P1= 4051.29 PID= 79091.0 SIGX= 0.0
T=0.2458 X=1.0809 XD= -152.80 XDD= 13261.1 P1= 4151.29 PID= 81121.0 SIGX= 0.0
T=0.2478 X=1.0194 XD= -125.51 XDD= 13461.1 P1= 4251.29 PID= 83151.0 SIGX= 0.0
T=0.2498 X=0.9599 XD= -98.22 XDD= 13661.1 P1= 4351.29 PID= 85181.0 SIGX= 0.0
T=0.2518 X=0.9024 XD= -70.93 XDD= 13861.1 P1= 4451.29 PID= 87211.0 SIGX= 0.0
T=0.2538 X=0.8469 XD= -43.64 XDD= 14061.1 P1= 4551.29 PID= 89241.0 SIGX= 0.0
T=0.2558 X=0.7934 XD= -16.35 XDD= 14261.1 P1= 4651.29 PID= 91271.0 SIGX= 0.0
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 DELA= 0.0256 DELAD= 0.0891 DELD=0.0073 DELDD=0.0543 SIGT= 75056.6 SIGAX= 35580.1 SIGR= 35580.1 SIGE= 48348.6
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 T=0.2577 X=10.070 XD= -104.08 XDD= 6848.4 P1= 4132.76 P10= 4132.76 P20= 23639.1 P3= 4815.22 P30= -54161.2
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 T=0.2597 X= 9.867 XD= -94.72 XDD= 6526.2 P1= 4158.62 P10= 4158.62 P20= 16652.1 P3= 4705.62 P30= -49872.9
 DELA= 0.0273 DELAD=0.0038 DELD=0.0077 DELDD=0.0316 SIGT= 80107.4 SIGAX= 38183.1 SIGR= 38183.1 SIGE= 51885.7
 T=0.2617 X= 9.691 XD= -81.97 XDD= 6235.7 P1= 4181.47 P10= 4181.47 P20= 5411.2 P3= 4606.78 P30= -44730.1
 DELA= 0.0275 DELAD= 0.1662 DELD=0.0078 DELDD=0.0017 SIGT= 80547.5 SIGAX= 38183.1 SIGR= 38183.1 SIGE= 51885.7
 T=0.2637 X= 9.539 XD= -69.77 XDD= 5979.5 P1= 4201.60 P10= 4201.60 P20= 1915.8 P3= 4519.50 P30= -38248.8
 DELA= 0.0276 DELAD= 0.1905 DELD=0.0078 DELDD=0.0080 SIGT= 80935.3 SIGAX= 38366.9 SIGR= 38366.9 SIGE= 52135.5
 T=0.2657 X= 9.411 XD= -58.04 XDD= 5759.9 P1= 4218.65 P10= 4218.65 P20= 7372.9 P3= 4444.32 P30= -31179.8
 DELA= 0.0277 DELAD= 0.0501 DELD=0.0079 DELDD=0.0114 SIGT= 81263.7 SIGAX= 38522.6 SIGR= 38522.6 SIGE= 52347.0
 T=0.2677 X= 9.307 XD= -46.72 XDD= 5577.8 P1= 4232.31 P10= 4232.31 P20= 11918.9 P3= 4381.62 P30= -24717.6
 DELA= 0.0278 DELAD=0.0719 DELD=0.0079 DELDD=0.0289 SIGT= 81526.8 SIGAX= 38742.6 SIGR= 38742.6 SIGE= 52646.0
 T=0.2697 X= 9.224 XD= -35.71 XDD= 5433.1 P1= 4242.75 P10= 4242.75 P20= 8801.1 P3= 4331.65 P30= -19273.0
 DELA= 0.0279 DELAD=0.0486 DELD=0.0079 DELDD=0.0205 SIGT= 81727.9 SIGAX= 38847.3 SIGR= 38847.3 SIGE= 52741.7
 T=0.2717 X= 9.163 XD= -24.96 XDD= 5325.7 P1= 4250.46 P10= 4250.46 P20= 1194.6 P3= 4294.59 P30= -14137.5
 DELA= 0.0279 DELAD= 0.0594 DELD=0.0079 DELDD=0.0022 SIGT= 81876.5 SIGAX= 38847.3 SIGR= 38847.3 SIGE= 52741.7
 T=0.2737 X= 9.124 XD= -14.39 XDD= 5256.1 P1= 4255.70 P10= 4255.70 P20= -2906.9 P3= 4270.48 P30= -8045.8
 DELA= 0.0279 DELAD= 0.1005 DELD=0.0079 DELDD=0.0115 SIGT= 81977.4 SIGAX= 38860.9 SIGR= 38860.9 SIGE= 52806.8
 T=0.2757 X= 9.106 XD= -13.92 XDD= 5224.8 P1= 4258.27 P10= 4258.27 P20= -794.1 P3= 4259.36 P30= -1981.9
 DELA= 0.0280 DELAD= 0.0270 DELD=0.0079 DELDD=0.0024 SIGT= 82026.8 SIGAX= 38984.3 SIGR= 38984.3 SIGE= 52838.6
 T=0.2777 X= 9.107 XD= 3.40 XDD= 3039.2 P1= 4258.16 P10= 4258.16 P20= 2859.3 P3= 4257.13 P30= 1835.3
 DELA= 0.0280 DELAD=0.0651 DELD=0.0079 DELDD=0.0092 SIGT= 82024.7 SIGAX= 38863.4 SIGR= 38863.4 SIGE= 52837.2
 T=0.2797 X= 9.120 XD= 9.87 XDD= 3027.3 P1= 4256.32 P10= 4256.32 P20= 1695.5 P3= 4249.93 P30= -2755.8
 DELA= 0.0280 DELAD=0.0597 DELD=0.0079 DELDD=0.0052 SIGT= 81989.4 SIGAX= 38866.6 SIGR= 38866.6 SIGE= 52814.5
 T=0.2817 X= 9.146 XD= 15.90 XDD= 3001.3 P1= 4253.16 P10= 4253.16 P20= -2738.6 P3= 4236.98 P30= -7635.3
 DELA= 0.0279 DELAD= 0.0060 DELD=0.0079 DELDD=0.0061 SIGT= 81928.4 SIGAX= 38837.7 SIGR= 38837.7 SIGE= 52775.2
 T=0.2837 X= 9.184 XD= 21.86 XDD= 2962.3 P1= 4248.82 P10= 4248.82 P20= -5259.2 P3= 4218.42 P30= -9977.8
 DELA= 0.0279 DELAD= 0.0355 DELD=0.0079 DELDD=0.0125 SIGT= 81844.8 SIGAX= 38798.0 SIGR= 38798.0 SIGE= 52721.3
 T=0.2857 X= 9.233 XD= 27.74 XDD= 2912.6 P1= 4243.18 P10= 4243.18 P20= -4294.4 P3= 4194.60 P30= -11567.6
 DELA= 0.0279 DELAD= 0.0007 DELD=0.0079 DELDD=0.0008 SIGT= 81736.3 SIGAX= 38746.6 SIGR= 38746.6 SIGE= 52651.4
 T=0.2877 X= 9.295 XD= 33.51 XDD= 2853.8 P1= 4236.10 P10= 4236.10 P20= -2356.5 P3= 4166.08 P30= -13503.1
 DELA= 0.0278 DELAD=0.0523 DELD=0.0079 DELDD=0.0015 SIGT= 81599.9 SIGAX= 36681.9 SIGR= 36681.9 SIGE= 52563.6
 T=0.2897 X= 9.367 XD= 39.15 XDD= 2787.1 P1= 4227.55 P10= 4227.55 P20= -2603.9 P3= 4133.50 P30= -15803.2
 DELA= 0.0278 DELAD=0.0642 DELD=0.0079 DELDD=0.0006 SIGT= 81435.2 SIGAX= 36603.9 SIGR= 36603.9 SIGE= 52457.5
 T=0.2917 X= 9.451 XD= 44.65 XDD= 2713.9 P1= 4217.71 P10= 4217.71 P20= -5189.7 P3= 4097.54 P30= -17814.0
 DELA= 0.0277 DELAD=0.0310 DELD=0.0075 DELDD=0.0089 SIGT= 81245.6 SIGAX= 38514.0 SIGR= 38514.0 SIGE= 52335.4

T=0.2937 X= 9.546 XD= 49.99 XDD= 2635.8 P1= 4206.71 P1D= 0.0 P2= 0.0 P2= 4206.71 P2D= -7459.6 P3= 4058.81 P3D= -19100.5
DELA= 0.0276 DELAD= -0.0044 DELD= -0.0078 DELD= 0.0146 SIGT= 81033.7 SIGAX= -4206.7 SIGE= 52198.8
T=0.2937 X= 9.651 XD= 55.18 XDD= 2554.3 P1= 4194.54 P1D= 0.0 P2= 0.0 P2= 4194.54 P2D= -7466.2 P3= 4017.91 P3D= -19824.0
DELA= 0.0275 DELAD= -0.0196 DELD= -0.0078 DELD= 0.0135 SIGT= 80799.2 SIGAX= 38302.4 SIGE= -4194.5 SIGE= 52047.8
T=0.2977 X= 9.766 XD= 60.21 XDD= 2471.3 P1= 4181.12 P1D= 0.0 P2= 0.0 P2= 4181.12 P2D= -6372.4 P3= 3975.48 P3D= -20402.3
DELA= 0.0275 DELAD= -0.0549 DELD= -0.0078 DELD= 0.0091 SIGT= 80540.7 SIGAX= 38179.9 SIGE= -4181.1 SIGE= 51881.3
T=0.2997 X= 9.892 XD= 65.06 XDD= 2387.7 P1= 4164.45 P1D= 0.0 P2= 0.0 P2= 4164.45 P2D= -6232.2 P3= 3932.04 P3D= -20947.9
DELA= 0.0274 DELAD= -0.0715 DELD= -0.0078 DELD= 0.0082 SIGT= 80258.1 SIGAX= 38045.9 SIGE= -4164.4 SIGE= 51699.2
T=0.3017 X= 10.026 XD= 69.75 XDD= 2304.7 P1= 4150.60 P1D= 0.0 P2= 0.0 P2= 4150.60 P2D= -7818.2 P3= 3888.06 P3D= -21342.3
DELA= 0.0273 DELAD= -0.0559 DELD= -0.0077 DELD= 0.0119 SIGT= 79952.9 SIGAX= 37901.2 SIGE= -4150.6 SIGE= 51502.6
T=0.3037 X= 10.171 XD= 74.28 XDD= 2223.1 P1= 4133.73 P1D= 0.0 P2= 0.0 P2= 4133.73 P2D= -9658.6 P3= 3843.99 P3D= -21412.4
DELA= 0.0272 DELAD= -0.0350 DELD= -0.0077 DELD= 0.0164 SIGT= 79627.9 SIGAX= 37747.1 SIGE= -4133.7 SIGE= 51293.2
T=0.3057 X= 10.323 XD= 79.64 XDD= 2143.2 P1= 4115.44 P1D= 0.0 P2= 0.0 P2= 4115.44 P2D= -9995.9 P3= 3800.12 P3D= -21194.7
DELA= 0.0270 DELAD= -0.0416 DELD= -0.0077 DELD= 0.0169 SIGT= 79283.6 SIGAX= 37583.9 SIGE= -4115.4 SIGE= 51071.5
T=0.3077 X= 10.485 XD= 82.45 XDD= 2067.7 P1= 4096.95 P1D= 0.0 P2= 0.0 P2= 4096.95 P2D= -9456.7 P3= 3756.72 P3D= -20878.6
DELA= 0.0269 DELAD= -0.0646 DELD= -0.0076 DELD= 0.0140 SIGT= 78919.3 SIGAX= 37411.2 SIGE= -4096.9 SIGE= 50836.8
T=0.3097 X= 10.655 XD= 86.91 XDD= 1995.0 P1= 4077.02 P1D= 0.0 P2= 0.0 P2= 4077.02 P2D= -9412.9 P3= 3714.03 P3D= -20582.7
DELA= 0.0268 DELAD= -0.0771 DELD= -0.0076 DELD= 0.0136 SIGT= 78535.6 SIGAX= 37229.3 SIGE= -4077.0 SIGE= 50589.7
T=0.3117 X= 10.832 XD= 90.43 XDD= 1925.9 P1= 4056.13 P1D= 0.0 P2= 0.0 P2= 4056.13 P2D= -10371.7 P3= 3672.16 P3D= -20288.2
DELA= 0.0266 DELAD= -0.0707 DELD= -0.0075 DELD= 0.0168 SIGT= 78133.1 SIGAX= 37034.5 SIGE= -4056.1 SIGE= 50330.4
T=0.3137 X= 11.018 XD= 94.61 XDD= 1868.6 P1= 4034.35 P1D= 0.0 P2= 0.0 P2= 4034.35 P2D= -11623.4 P3= 3631.21 P3D= -19898.8
DELA= 0.0265 DELAD= -0.0587 DELD= -0.0075 DELD= 0.0184 SIGT= 77713.4 SIGAX= 36839.6 SIGE= -4034.3 SIGE= 50060.1
T=0.3157 X= 11.211 XD= 98.27 XDD= 1799.2 P1= 4011.71 P1D= 0.0 P2= 0.0 P2= 4011.71 P2D= -12132.9 P3= 3591.20 P3D= -19417.3
DELA= 0.0263 DELAD= -0.0598 DELD= -0.0075 DELD= 0.0199 SIGT= 77277.5 SIGAX= 36632.9 SIGE= -4011.7 SIGE= 49779.2
T=0.3177 X= 11.411 XD= 101.81 XDD= 1741.8 P1= 3988.23 P1D= 0.0 P2= 0.0 P2= 3988.23 P2D= -11919.8 P3= 3552.16 P3D= -18925.2
DELA= 0.0262 DELAD= -0.0744 DELD= -0.0074 DELD= 0.0183 SIGT= 76825.1 SIGAX= 36416.5 SIGE= -3988.2 SIGE= 49487.8
T=0.3197 X= 11.618 XD= 105.24 XDD= 1688.2 P1= 3963.88 P1D= 0.0 P2= 0.0 P2= 3963.88 P2D= -11838.6 P3= 3514.06 P3D= -18514.5
DELA= 0.0260 DELAD= -0.0861 DELD= -0.0074 DELD= 0.0174 SIGT= 76356.0 SIGAX= 36196.1 SIGE= -3963.9 SIGE= 49185.7
T=0.3217 X= 11.832 XD= 108.56 XDD= 1634.2 P1= 3938.71 P1D= 0.0 P2= 0.0 P2= 3938.71 P2D= -12513.1 P3= 3476.88 P3D= -18159.4
DELA= 0.0259 DELAD= -0.0832 DELD= -0.0073 DELD= 0.0214 SIGT= 75871.2 SIGAX= 35986.5 SIGE= -3938.7 SIGE= 48873.4
T=0.3237 X= 12.052 XD= 111.79 XDD= 1591.7 P1= 3912.79 P1D= 0.0 P2= 0.0 P2= 3912.79 P2D= -13468.2 P3= 3440.57 P3D= -17798.9
DELA= 0.0257 DELAD= -0.0742 DELD= -0.0073 DELD= 0.0188 SIGT= 75372.0 SIGAX= 35729.7 SIGE= -3912.8 SIGE= 48551.8
T=0.3257 X= 12.279 XD= 114.93 XDD= 1548.5 P1= 3888.82 P1D= 0.0 P2= 0.0 P2= 3888.82 P2D= -13859.3 P3= 3405.05 P3D= -17413.1
DELA= 0.0255 DELAD= -0.0762 DELD= -0.0072 DELD= 0.0221 SIGT= 74859.1 SIGAX= 35486.5 SIGE= -3888.8 SIGE= 48221.4
T=0.3277 X= 12.512 XD= 117.98 XDD= 1508.3 P1= 3868.82 P1D= 0.0 P2= 0.0 P2= 3868.82 P2D= -13747.0 P3= 3370.25 P3D= -17062.6
DELA= 0.0253 DELAD= -0.0872 DELD= -0.0072 DELD= 0.0213 SIGT= 74332.2 SIGAX= 35236.8 SIGE= -3868.8 SIGE= 47882.0
T=0.3297 X= 12.751 XD= 120.96 XDD= 1471.0 P1= 3838.75 P1D= 0.0 P2= 0.0 P2= 3838.75 P2D= -13798.3 P3= 3336.11 P3D= -16790.9
DELA= 0.0252 DELAD= -0.0951 DELD= -0.0071 DELD= 0.0203 SIGT= 73791.5 SIGAX= 34980.4 SIGE= -3830.7 SIGE= 47533.7
T=0.3317 X= 12.995 XD= 123.87 XDD= 1436.4 P1= 3801.98 P1D= 0.0 P2= 0.0 P2= 3801.98 P2D= -14362.2 P3= 3302.54 P3D= -16584.2
DELA= 0.0250 DELAD= -0.0923 DELD= -0.0071 DELD= 0.0222 SIGT= 73237.4 SIGAX= 34717.8 SIGE= -3802.0 SIGE= 47176.7
T=0.3337 X= 13.246 XD= 126.70 XDD= 1404.0 P1= 3772.58 P1D= 0.0 P2= 0.0 P2= 3772.58 P2D= -15124.9 P3= 3269.48 P3D= -16386.5
DELA= 0.0248 DELAD= -0.0860 DELD= -0.0070 DELD= 0.0244 SIGT= 72671.1 SIGAX= 34449.3 SIGE= -3772.6 SIGE= 46812.0
T=0.3357 X= 13.502 XD= 129.48 XDD= 1373.9 P1= 3742.57 P1D= 0.0 P2= 0.0 P2= 3742.57 P2D= -15370.7 P3= 3236.84 P3D= -16182.0
DELA= 0.0246 DELAD= -0.0888 DELD= -0.0070 DELD= 0.0244 SIGT= 72093.0 SIGAX= 34175.2 SIGE= -3742.6 SIGE= 46439.6
T=0.3377 X= 13.764 XD= 132.20 XDD= 1345.7 P1= 3711.94 P1D= 0.0 P2= 0.0 P2= 3711.94 P2D= -15396.9 P3= 3204.56 P3D= -16019.3
DELA= 0.0244 DELAD= -0.0968 DELD= -0.0069 DELD= 0.0232 SIGT= 71502.9 SIGAX= 33895.5 SIGE= -3711.9 SIGE= 46059.5
T=0.3397 X= 14.031 XD= 134.86 XDD= 1294.4 P1= 3680.69 P1D= 0.0 P2= 0.0 P2= 3680.69 P2D= -15500.0 P3= 3172.56 P3D= -15912.8
DELA= 0.0242 DELAD= -0.1019 DELD= -0.0069 DELD= 0.0236 SIGT= 70901.0 SIGAX= 33610.2 SIGE= -3680.7 SIGE= 45671.7
T=0.3417 X= 14.303 XD= 137.47 XDD= 1259.4 P1= 3648.85 P1D= 0.0 P2= 0.0 P2= 3648.85 P2D= -16002.9 P3= 3140.80 P3D= -15857.0
DELA= 0.0240 DELAD= -0.0992 DELD= -0.0068 DELD= 0.0251 SIGT= 70287.6 SIGAX= 33319.4 SIGE= -3648.8 SIGE= 45276.6
T=0.3437 X= 14.581 XD= 140.04 XDD= 1227.6 P1= 3616.46 P1D= 0.0 P2= 0.0 P2= 3616.46 P2D= -16552.1 P3= 3109.20 P3D= -15800.9
DELA= 0.0238 DELAD= -0.1060 DELD= -0.0067 DELD= 0.0261 SIGT= 69663.7 SIGAX= 33023.7 SIGE= -3616.5 SIGE= 44874.7
T=0.3457 X= 14.863 XD= 142.56 XDD= 1194.7 P1= 3584.53 P1D= 0.0 P2= 0.0 P2= 3584.53 P2D= -16726.3 P3= 3077.72 P3D= -15747.7
DELA= 0.0236 DELAD= -0.0998 DELD= -0.0066 DELD= 0.0261 SIGT= 69029.4 SIGAX= 32723.0 SIGE= -3584.5 SIGE= 44466.2
T=0.3477 X= 15.151 XD= 145.03 XDD= 1169.4 P1= 3550.06 P1D= 0.0 P2= 0.0 P2= 3550.06 P2D= -16935.8 P3= 3046.30 P3D= -15729.1
DELA= 0.0234 DELAD= -0.1089 DELD= -0.0065 DELD= 0.0274 SIGT= 68384.7 SIGAX= 32417.3 SIGE= -3550.1 SIGE= 44050.8
T=0.3497 X= 15.443 XD= 147.47 XDD= 1148.1 P1= 3516.05 P1D= 0.0 P2= 0.0 P2= 3516.05 P2D= -17362.0 P3= 2983.47 P3D= -15791.9
DELA= 0.0231 DELAD= -0.1068 DELD= -0.0065 DELD= 0.0271 SIGT= 67729.6 SIGAX= 32106.8 SIGE= -3516.1 SIGE= 43628.9
T=0.3517 X= 15.741 XD= 149.86 XDD= 1128.1 P1= 3481.54 P1D= 0.0 P2= 0.0 P2= 3481.54 P2D= -17742.2 P3= 2951.99 P3D= -15800.6
DELA= 0.0229 DELAD= -0.1068 DELD= -0.0065 DELD= 0.0274 SIGT= 67064.7 SIGAX= 31791.6 SIGE= -3481.5 SIGE= 43200.5
T=0.3537 X= 16.043 XD= 152.22 XDD= 1109.4 P1= 3446.53 P1D= 0.0 P2= 0.0 P2= 3446.53 P2D= -17742.2 P3= 2951.99 P3D= -15800.6
DELA= 0.0226 DELAD= -0.1059 DELD= -0.0064 DELD= 0.0274 SIGT= 66390.4 SIGAX= 31472.0 SIGE= -3446.5 SIGE= 42766.2
T=0.3557 X= 16.349 XD= 154.54 XDD= 1151.3 P1= 3411.05 P1D= 0.0 P2= 0.0 P2= 3411.05 P2D= -17965.1 P3= 2920.41 P3D= -15892.1
DELA= 0.0224 DELAD= -0.1073 DELD= -0.0063 DELD= 0.0285 SIGT= 65707.0 SIGAX= 31144.0 SIGE= -3411.1 SIGE= 42325.9
T=0.3577 X= 16.660 XD= 156.82 XDD= 1133.8 P1= 3375.10 P1D= 0.0 P2= 0.0 P2= 3375.10 P2D= -17997.6 P3= 2888.59 P3D= -15959.7

DELA= 0.0222 DELAD=-0.1130 DELD=-0.0063 DELDD= 0.0276 SIGT= 65014.4 SIGAX= 30819.7 SIGR= -3375.1 SIGE= 41879.8
 T=0.3596 X=16.976 XD= 159.07 XDD= 1116.6 PI= 3338.67 PID= 3338.67 P2D= -18204.3 P3= 2856.84 P3D= -16054.0
 DELA= 0.0219 DELAD=-0.1144 DELD=-0.0062 DELDD= 0.0285 SIGT= 64312.7 SIGAX= 30487.0 SIGR= -3338.7 SIGE= 41427.8
 T=0.3616 X=17.296 XD= 161.29 XDD= 1099.8 PI= 3301.80 PID= 3301.80 P2D= -18581.0 P3= 2824.81 P3D= -16160.1
 DELA= 0.0217 DELAD=-0.1128 DELD=-0.0061 DELDD= 0.0292 SIGT= 63602.4 SIGAX= 30150.3 SIGR= -3301.8 SIGE= 40970.3
 T=0.3636 X=17.620 XD= 163.47 XDD= 1083.3 PI= 3264.50 PID= 3264.50 P2D= -18870.2 P3= 2792.59 P3D= -16255.7
 DELA= 0.0214 DELAD=-0.1130 DELD=-0.0061 DELDD= 0.0293 SIGT= 62883.9 SIGAX= 29809.7 SIGR= -3264.5 SIGE= 40507.5
 T=0.3656 X=17.949 XD= 165.62 XDD= 1067.0 PI= 3226.78 PID= 3226.78 P2D= -18913.6 P3= 2760.16 P3D= -16356.5
 DELA= 0.0212 DELAD=-0.1174 DELD=-0.0060 DELDD= 0.0293 SIGT= 62157.3 SIGAX= 29465.3 SIGR= -3226.8 SIGE= 40039.4
 T=0.3676 X=18.282 XD= 167.74 XDD= 1050.8 PI= 3188.63 PID= 3188.63 P2D= -19025.4 P3= 2727.51 P3D= -16477.6
 DELA= 0.0209 DELAD=-0.1205 DELD=-0.0059 DELDD= 0.0294 SIGT= 61422.5 SIGAX= 29117.0 SIGR= -3188.6 SIGE= 39566.1
 T=0.3696 X=18.619 XD= 169.82 XDD= 1034.8 PI= 3150.07 PID= 3150.07 P2D= -19320.0 P3= 2694.62 P3D= -16616.7
 DELA= 0.0207 DELAD=-0.1196 DELD=-0.0059 DELDD= 0.0302 SIGT= 60679.8 SIGAX= 28764.9 SIGR= -3150.1 SIGE= 39087.6
 T=0.3716 X=18.961 XD= 171.87 XDD= 1018.8 PI= 3111.13 PID= 3111.13 P2D= -19678.7 P3= 2661.49 P3D= -16749.4
 DELA= 0.0204 DELAD=-0.1176 DELD=-0.0058 DELDD= 0.0309 SIGT= 59929.7 SIGAX= 28409.3 SIGR= -3111.1 SIGE= 38604.4
 T=0.3736 X=19.306 XD= 173.90 XDD= 1002.8 PI= 3071.82 PID= 3071.82 P2D= -19757.6 P3= 2628.09 P3D= -16869.4
 DELA= 0.0202 DELAD=-0.1212 DELD=-0.0057 DELDD= 0.0305 SIGT= 59172.4 SIGAX= 28050.3 SIGR= -3071.8 SIGE= 38116.6
 T=0.3756 X=19.656 XD= 175.88 XDD= 986.8 PI= 3032.13 PID= 3032.13 P2D= -19839.5 P3= 2594.43 P3D= -17008.7
 DELA= 0.0199 DELAD=-0.1240 DELD=-0.0056 DELDD= 0.0310 SIGT= 58407.7 SIGAX= 27687.8 SIGR= -3032.1 SIGE= 37624.1
 EC= -348.67212 EF= -78452.3750

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